

# Benchmark on the Aerodynamics of a Rectangular 5:1 Cylinder: an overview after the first four years of activity

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## Abstract

In July 2008, a benchmark study on the aerodynamics of a stationary rectangular cylinder with chord-to-depth ratio equal to 5 (BARC) was launched. This paper gives an outline of the state of the art on the aerodynamics of 5:1 rectangular cylinders prior to the starting of BARC, and summarizes the results obtained by the contributors during the first four years of activity. The results of about 70 realisations of the BARC flow configuration obtained under a nominally common set-up in both wind tunnel experiments and numerical simulations are compared among themselves and with the data available in the literature prior to BARC, in terms of bulk parameters, flow and aerodynamic load statistics, pressure and force spanwise correlations. It is shown that the near wake flow, the base pressure and, hence, the drag coefficient obtained in the different flow realisations are in very good agreement. Conversely, the flow features along the cylinder lateral surfaces and, hence, the lift are strongly sensitive to set-up and modelling, leading to a significant dispersion of both wind tunnel measurements and numerical predictions. Finally, a possible asymmetry of the time averaged flow has been recognised both in wind tunnel tests and in numerical simulations.

*Key words:* Bluff-body aerodynamics, BARC benchmark, numerical simulations, wind tunnel tests

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## 1 Introduction

In July 2008, a benchmark study on the aerodynamics of rectangular cylinders (a Benchmark on the Aerodynamics of a Rectangular 5:1 Cylinder, BARC) was announced during the VI Colloquium on Bluff Body Aerodynamics and Applications (BBAA VI). This paper aims at providing a progress report of the BARC activities four years after its announcement.

BARC is aimed at establishing a platform for discussion among scientists working on bluff body aerodynamics, and in particular it concerns the analysis of the turbulent, separated flow around an elongated rectangular cylinder. The characteristics of the flow field around rectangular bodies is of great interest both for fundamental research and for applications. From the fundamental research point of view, in spite of the simple and nominally two-dimensional geometry, the flow over an elongated rectangular cylinder at high Reynolds numbers is highly complex, being three dimensional, turbulent and characterized by unsteady flow separation and reattachment. The main flow features of the BARC configuration are described in more detail in the following. On the other hand, thanks to the simple geometry, a detailed analysis of the flow dynamics can be carried out, and different patterns, which can also be found when dealing with more complex geometries, can be identified. As for applications, this benchmark problem provides useful information on the aerodynamics of a wide range of bluff bodies of interest in civil engineering (e.g. long-span bridge decks or high-rise buildings) as well as in other engineering areas.

It is well known that two-dimensional(2D) rectangular cylinders are characterized by one single geometric parameter, i.e. the ratio of the alongwind dimension (Breadth) to the crosswind dimension (Depth),  $B/D$ , which governs their aerodynamic behaviour (see e.g. [Nakaguchi et al. \[1968\]](#), [Stokes and Welsh \[1986\]](#)). Other geometric (e.g. surface roughness, corner sharpness) and flow parameters (e.g. Reynolds number, turbulence intensity and scale) play a major or minor role, depending on the  $B/D$  value and on their range of variability. For small  $B/D$  ratios ( $< 2.5$ ), the flow separates from the leading edge and does not reattach to the side faces of the cylinder, with vortex shedding occurring only from the leading edge. For larger  $B/D$  ratios, the shear layer impinges on the side faces of the cylinder. For moderate  $B/D$  ratios (roughly between 2.5 and 3.5), reattachment is intermittent, and vortex shedding still occurs only from the leading edge. For  $B/D$  ratios greater than 3.5, reattachment is permanent, and vortex shedding occurs from both the leading and the trailing edges. Under this circumstance, the flow patterns depends on  $B/D$  in a discontinuous fashion. Trailing edge shedding is influenced by the dynamics of the shear layer detaching at the upstream corners and impinging on the side face downstream of the separation bubble; the breadth-based Strouhal number is a multiple of 0.55 – 0.60, depending on the number of vortices simultaneously attached to each face of the cylinder (see e.g. [Nakamura](#)

et al. [1991], Stokes and Welsh [1986]). For  $B/D < 6$  the Strouhal number is  $1 \times (0.55 - 0.60)$ , for  $6 < B/D < 9$  the Strouhal number is  $2 \times (0.55 - 0.60)$ , for  $9 < B/D < 12$  it is  $3 \times (0.55 - 0.60)$ .

It follows that the choice made for the BARC of a  $B/D$  ratio of 5 brings a completely reattached flow ( $B/D > 3.5$ ) and simultaneously one single attached vortex on each face of the cylinder ( $B/D < 6$ ).

In the following the studies published prior to the BARC announcement are briefly reviewed, in order to provide the context in which the BARC benchmark was launched. The overview is limited to the aerodynamics of rectangular cylinders with  $B/D$  ratio equal to 5 or close to it, i.e.  $3.5 \leq B/D \leq 6$ . Bearing in mind that BARC addresses both the wind tunnel and the computational approaches, these are discussed separately.

The reviewed wind/water tunnel studies are chronologically listed in Table 1. For the sake of clarity, the listed references can be grouped in two subsets

Table 1

Wind/water tunnel studies around rectangular cylinders prior to BARC.  $B$  is the cylinder breadth,  $D$  the depth and  $L$  the spanwise length;  $I_u$  is the turbulence intensity of the oncoming flow.

Authors	$B/D$	$Re_D$	$L/D$	Blockage (%)	$I_u$ (%)
Nakamura and Yoshimura [1982]	0.2-5	5500-140000	10;25	2;5	0.1
Okajima [1982]	1-4	200-20000	13.3-100	< 2.5	< 0.5%
Okajima et al. [1983]	1-9	42000	—	1	0.4
Parker and Welsh [1983]	< 52	15000-31000	203	0.25	0.2
Stokes and Welsh [1986]	< 16	8000-44300	203-407	0.25-0.49	0.2
Nakamura and Nakshima [1986]	1-10	2500-300000	3-30	$\leq 6$	—
Nakamura et al. [1991]	3-16	1000-3000	100	0.17	0.3
Matsumoto et al. [2003]	5	—	15	3	—
Ricciardelli and Marra [2008]	5	63600	38.7	3.5	$\simeq 1$
Haan and Kareem [2009]	6.7	130000-320000	26.5		0.4–11.6
Le et al. [2009]	1;5	18000-54000	10.4	—	9.5-11.5
Ricciardelli [2010]	5	64000	38.7	3.5	$\simeq 1$

by referring to both the chronological and thematic criteria. The studies developed in the eighties and nineties mainly investigate the variability of the aerodynamic behaviour versus the  $B/D$  ratio by means of force and/or velocity measurements. The more recent works focus their attention on the ratio  $B/D = 5$  and on the spanwise coherence of the flow by means of pressure measurements over the cylinder surface.

From the results of wind and water tunnel experiments on rectangular prisms with  $B/D = 1, 2, 3$  and 4, Okajima [1982] investigated the variation of the Strouhal number and flow characteristics with the Reynolds number in the range of 70 to 20,000. He concluded that the minimum aspect ratio giving rise to flow reattachment is Reynolds number dependent, and tends to 2.8 at high Reynolds. Conversely, the minimum Reynolds number giving rise to fully separated flow depends on the aspect ratio. In Okajima et al. [1983] the

variation of the drag and lift coefficients and of the Strouhal number with the aspect ratio was investigated, and the existence of an unsteady reattachment of the separated shear layer on the side surfaces for values of the aspect ratio in the range of 2.0 to 2.8 was pointed out.

[Parker and Welsh \[1983\]](#) and [Stokes and Welsh \[1986\]](#) analysed the effects that the application of sound has on the characteristics of the shear layer and of the separation bubble of rectangular cylinders with  $B/D$  ratios up to 52.

[Nakamura and Nakshima \[1986\]](#) suggested that for rectangular cylinders with  $B/D$  in the range of 3 to 15 the vortex shedding mechanism is not of Karman type, i.e. triggered directly by the interaction between upper and lower shear layers, but it is due to the impinging shear layer instability. They also showed that shear layer instability, vortex shedding and vortex excitation were possible also in the presence of a splitter plate, and observed that vortex excitation was possible for elastically suspended models for which hot-wire measurements had not shown sharp spectra at rest. Successively, in [Nakamura et al. \[1991\]](#) the characteristics of vortex shedding from rectangular cylinders with  $3 < B/D < 13$  was investigated, and the discontinuous variation of the Strouhal number earlier pointed out was found.

In [Matsumoto et al. \[2003\]](#) the spanwise coherence of pressure fluctuations on a rectangular cylinder with  $B/D = 5$  was investigated through wind tunnel tests in smooth and turbulent flow. It was found that: (i) the pressure coherence is higher than that of the longitudinal component of the oncoming turbulence; (ii) the pressure fluctuation slightly upstream the reattaching point plays the most significant role in the evaluation of buffeting force.

[Ricciardelli and Marra \[2008\]](#) discussed the sectional distribution of the statistics (mean, standard deviation, skewness and kurtosis) of the pressure coefficients on a rectangular stationary and vibrating cylinder with  $B/D = 5$ , as derived from wind tunnel experiments. In addition the spanwise correlation of pressure fluctuations was also discussed, in the cases of smooth and turbulent incoming flow. In [Ricciardelli \[2010\]](#), the effects of the vibration regime on the spanwise correlation of aerodynamic forces and of stagnation and base pressure for a rectangular cylinder with  $B/D = 5$  were discussed, as derived from wind tunnel tests.

[Le et al. \[2009\]](#) analysed the results obtained by means of wind tunnel tests for a highly turbulent incoming flow ( $10\% \leq I_x \leq 12\%$ ) around a square and a rectangular 5:1 cylinder. First, statistics on chordwise pressure distribution were discussed. Second, Fourier and wavelet spectra of spanwise coherent turbulent structures and of pressure were analysed. The differences from the case of the fully separated and reattached flow were pointed out, and the larger coherence of pressures with respect to turbulent fluctuations was confirmed.

[Haan and Kareem \[2009\]](#) examined turbulence effects on the self-excited and buffeting forces on an oscillating rectangular prism having  $B/D = 6.67$ .

Computational studies on the aerodynamics of rectangular cylinders available prior to the announcement of BARC are chronologically listed in Table 2. Both 2D and three-dimensional (3D) features of the low-Reynolds number flow

Table 2

Computational studies around rectangular elongated cylinders previous to BARC.  $Re_D$  is the Reynolds number based on the cylinder depth and the freestream velocity; the remaining symbols are the same as in Table 1.

Authors	turb. mod.	$B/D$	$Re_D$	$L/D$
<a href="#">Tamura et al. [1993]</a>	no model	5	$10^4$	2
<a href="#">Tamura and Ito [1996]</a>	no model	$0.6 \leq B/D \leq 8$	$10^4$	2
<a href="#">Yu and Kareem [1998]</a>	LES	$1 \leq B/D \leq 4$	$10^5$	2
<a href="#">Shimada and Ishihara [2002]</a>	RANS	$0.6 \leq B/D \leq 8$	$2. \times 10^4$	0 (2D)

( $10^2 \leq Re_D \leq 10^3$ ),  $Re_D$  being the Reynolds number based on the cylinder depth, around rectangular cylinders have been clarified in several studies, e.g. [Nakamura et al. \[1996\]](#), [Ohya et al. \[1992\]](#), [Hourigan et al. \[2001\]](#), [Tan et al. \[2004\]](#). However, these Reynolds numbers being much lower than those of interest for BARC, these studies are not reviewed herein, and interested readers can refer to the cited papers and to their references. On the other hand, the high-Reynolds number flow conditions (i.e.  $Re \geq 10^4$ ) have been investigated using computational techniques with special emphasis on the dependence of the aerodynamic behaviour on the chord-to-depth ratio, e.g. in [Shimada and Ishihara \[2002\]](#), [Tamura and Ito \[1996\]](#), [Yu and Kareem \[1998\]](#). In particular, [Shimada and Ishihara \[2002\]](#) also investigated, among others, the rectangular 5:1 cylinder in a 2D domain with a two-layer modified  $k - \varepsilon$  RANS turbulence model. Their computations succeeded in reproducing a smooth and periodic vortex shedding also at high Reynolds numbers, as well as the discontinuity in the Strouhal number at  $B/D = 2.8$  and  $B/D = 6.0$ . Nonetheless, the pressure and force fluctuations were underestimated. The authors argued that this is due to Reynolds averaging of the Navier-Stokes equations. More recently, [Yu et al. \[2013\]](#) carried out a numerical parametric study of the flow around rectangular cylinder having  $B/D$  ranging from 0.3 to 7, by means of large-eddy simulations at  $Re_D = 10^5$ . They found the reattachment of the separated flow on the cylinder side faces for  $B/D$  from 3 and greater. The numerical results for aerodynamic forces and Strouhal numbers reproduced the trends observed in the experiments.

[Tamura and Ito \[1996\]](#) discretized the 3D Navier-Stokes equations through a finite-difference technique at  $Re = 10^4$  and studied the mechanism of vortex formation for several rectangular cylinders with different  $B/D$  ratios. To our knowledge, only [Tamura et al. \[1993\]](#) focused on the rectangular 5:1 cylinder by using a finite-difference discretisation of the Navier-Stokes equations on structured O-grids in 2D and 3D. A third-order upwind scheme was adopted for the convective terms and no turbulence model was employed. The separation bubble was recognized to be responsible for irregularly fluctuating pressure patterns around the reattachment area on the side surface. The dynamic characteristics of the shear layer separated from the leading edge and the in-

stability of the strong shear region to form the wake vortices were discussed by comparing two- and three-dimensional computed flows. In a successive work of [Tamura et al. \[1995\]](#) also the forced oscillating cylinder was studied at the same incoming flow conditions by means of a 2D model.

In this context, the BARC is a blind benchmark, which has not adopted a single set of measurements as a reference at its launching. Hence, the BARC differs, for instance, from the pioneering benchmark on the aerodynamics of the square cylinder [see for the review of the obtained results [Rodi , 1997](#), [Rodi, 2002](#), [Voke, 1997](#)], included since the Nineties in the ERCOFTAC classic collection database [Ercoftac test case LES2](#) and now in the ERCOFTAC QNET-CFD Knowledge Base Wiki [Ercoftac UFR2-02](#). Indeed, this benchmark used as a reference the measurements by [Lyn et al. \[1995\]](#). The lack of available reference data is a quite common situation in many engineering flow problems. One of the main goals of the BARC and of the present overview is to assess the reliability and the dispersion of the measurements or of the numerical predictions of different quantities of practical interest for a benchmark configuration which contains most of the difficulties also found in more realistic flows of engineering interest. This is accomplished by collecting as much as possible of experimental and numerical data produced by different groups, in different facilities or by means of different numerical models and codes, but all within a common set-up as a-priori specified. Clearly, both wind tunnel tests and computational simulations suffer from set-up uncertainties and from measure or modelling errors. The impact of these uncertainties on the measurements/predictions of the main flow quantities is characterized in this overview and, when possible, the effects of different set-up parameters are singled out and analyzed. This may also give information useful to assess the possibility of developing integrated procedures relying on both wind tunnel and computational outcomes aimed at improving the reliability of the results, and, eventually, to develop Best Practice Advice for wind tunnel tests and computational simulations. Finally, the BARC aims at making available at its maturity a statistic database, which is expected to be further enriched and updated by new realisations.

At present, 78 participants in 43 teams are registered to access the web site member area (<http://www.aniv-iawe.org/barc>). Most of the participants come from academia (70%) and from research centres (19%), with smaller percentages coming from the consultancy (9%) and the industry (2%).

During the first four years of activity, thematic sessions have been devoted to BARC at the 5th European and African Conference on Wind Engineering (EACWE, 2009, Florence), at the 5th International Symposium on Computational Wind Engineering (CWE, 2010, Chapel Hill) and at the 13th International Conference on Wind Engineering (ICWE, 2011, Amsterdam). The last thematic session hosted 9 contributions and a final synopsis and overview. Besides the contributions to the mentioned conference thematic sessions, four journal papers explicitly referring to BARC have been published up to now [Bruno et al. \[2010, 2012\]](#), [Mannini et al. \[2010, 2011\]](#).

Besides this introduction, the paper is organized into four more sections. In Section 2, the test case main set-up and the result format requested to the participant are recalled. In Section 3 the wind tunnel tests and the computational simulations performed up to now are described; their results are then compared in Section 4., which is further divided in three subsections: Sec. (4.1) deals with the analysis of the flow bulk parameters, the main flow features and statistics are described and compared in Sec. (4.2), while pressure and force correlations are reported in Sec. (4.3). Finally, in Section 5 some preliminary conclusions are attempted and further research perspectives are briefly sketched.

## 2 Test case main set-up and result format

As mentioned in the Introduction, BARC addresses the high Reynolds number, external, unsteady flow over a stationary, sharp-edged smooth rectangular cylinder, and the associated aerodynamic loads [Bartoli et al. \[2008a\]](#). The breadth ( $B$ ) to depth ( $D$ ) ratio is set equal to 5.

The following common requirements are set for both wind tunnel tests and numerical simulations:

- the depth-based Reynolds number  $Re_D = UD/\nu$  has to be in the range of  $2 \times 10^4$  to  $6 \times 10^4$ ;
- the incoming flow has to be set parallel to the breadth of the rectangle, i.e.  $\alpha = 0$ ,  $\alpha$  being the angle of attack;
- the maximum intensity of the longitudinal component of the freestream turbulence is set to  $I_x = 0.01$ ;
- the minimum spanwise length of the cylinder for wind tunnel tests and 3D numerical simulations is set to  $L/D = 3$ ;

The following additional requirements are specified for wind tunnel tests:

- the maximum acceptable radius of curvature of the edges of wind tunnel models is set to  $R/D = 0.05$ ;
- the maximum wind tunnel blockage is set to 5%;
- all the points of measurement have to be outside the boundary layers developed at the tunnel walls;
- uniformity of the flow at all measurement points must be checked in the empty tunnel and appropriately documented.

In addition to the main set-up described above, sensitivity studies are strongly encouraged. The following additional values of the parameters are suggested for both wind tunnel tests and numerical simulations:



- angles of incidence  $\alpha = 1, 3, 6$ ;
- Reynolds number  $\text{Re}_D = 10^3, 10^4, 10^5, 10^6$ ;
- turbulence intensity  $I_x = 0.02, 0.05, 0.10$ .

The flow quantities presented in the following are made nondimensional by using the undisturbed flow field velocity  $U$ , the cylinder depth  $D$  and the fluid density  $\rho$ , unless specified otherwise.

Data can be uploaded to the BARC website by registered participants. set-up information and output data requested for numerical simulations and wind tunnel tests are set in Requests for Computational Simulations [Bartoli et al. \[2008b\]](#) and Requests for Wind Tunnel Tests [Bartoli et al. \[2008c\]](#), respectively.

### 3 Description of the wind tunnel tests and computational studies

The Special Technical Session at ICWE13 hosted 9 original contributions, summarising the results obtained in 59 realisations of the flow, i.e. 18 wind tunnel tests in three different facilities [Bartoli et al. \[2011\]](#), [Bronkhorst et al. \[2011\]](#), [Shirato et al. \[2011\]](#), and 41 computational simulations [Arslan et al. \[2011\]](#), [Bruno et al. \[2011\]](#), [Grozescu et al. \[2011a\]](#), [Mannini and Schewe \[2011\]](#), [Ribeiro \[2011\]](#), [Wei and Kareem \[2011\]](#). The resulting database is complemented by the studies presented at the previous BARC thematic sessions [e.g. [Schewe, 2009](#), [Shirato et al., 2010](#)], and the ones published in the recent past on international journals [Bruno et al. \[2010, 2012\]](#), [Mannini et al. \[2010, 2011\]](#). Most of the studies adopt incoming flow characteristics in accordance with the range prescribed by the BARC main set-up (see Sect. 2) and/or with the ones suggested for the sensitivity studies [e.g. turbulence intensity and length scale in [Shirato et al., 2010, 2011](#)]. The adopted incoming flow features are summarized in Figure 1. As for the freestream turbulence intensity, all the

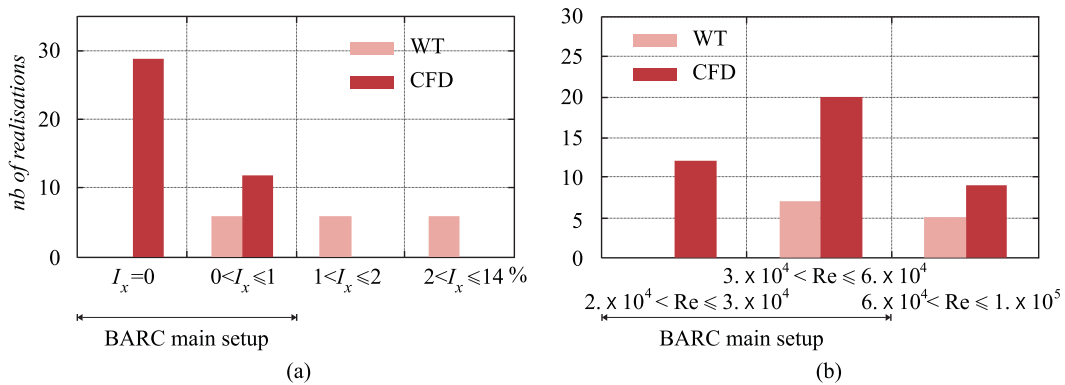


Fig. 1. Adopted incoming flow features: longitudinal turbulence intensity (a) and Reynolds number (b)

computational studies using Large-Eddy Simulation (LES) or Detached Eddy



Simulation (DES) turbulence approaches adopt perfectly smooth incoming flow, mainly because of the difficulties involved in the generation of realistic incoming turbulence features within these approaches. Conversely, perfectly smooth flow conditions cannot be obtained in wind tunnels, where a residual turbulence always exists; on the other hand, grid generated turbulence is easy to obtain in wind tunnel tests. Hence, the mentioned differences among computational and wind tunnel approaches do not allow to compare flowfields obtained exactly in the same conditions, but the complementary features of each approach allow the effects of incoming turbulence to be investigated in a collaborative framework. Figure 1b also shows that another parameter significantly varying among the different contributions is the freestream Reynolds number, even if it keeps the same order of magnitude and most of the values fall in the range specified in the BARC main set-up.

In the following subsections, the set-up and objectives of both wind tunnel tests and computational simulations are shortly summarized. The given synopsis is intended to point out common and/or complementary features of the contributions to BARC rather than to detail each single study, for which interested readers are addressed to the original papers.

### 3.1 *Wind tunnel tests*

The main characteristics of the set-ups of the wind tunnel contributions to BARC are given in table 3. All the experimental contributions are aimed at reproducing as much as possible an unconfined 2D nominal flow, as in the requirements of the BARC benchmark: the ratio between the cylinder spanwise length,  $L$  and  $B$  ranges from 3 to 10.8, i.e.  $L/D = 15 - 54$ , and the blockage is generally lower than 2%, except for one case in Shirato et al. [2011]. Three of the experimental contributions Bartoli et al. [2011], Bronkhorst et al. [2011], Schewe [2009] are aimed at obtaining measurements for freestream conditions that are as smooth as possible. A different approach is that of the paper of Shirato et al. (2011), in which the characteristics of the spanwise coherence of the aerodynamic action of rectangular cylinders with  $B/D$  ratios ranging from 2.2 to 10 is investigated, for different intensity and scale of the incoming turbulence, obtained by means of three grid arrangements placed upstream the model.

As for reduced scale models, it is worth pointing out that the tests performed by Bartoli et al. [2011] are characterised by an ad hoc conceived, aluminium model shaped through a countersink procedure in order to obtain very accurate prismatic shape and sharp edges. The model adopted by Bronkhorst et al. [2011] consists of plastic and wooden parts including interchangeable edge elements with radii of curvature of  $R/D = 0, 0.01, 0.02$  and  $0.5$  to investigate the influence of edge sharpness on the flow. The model in Schewe [2009] was originally used as a generic H-shaped section in a previous investigation and it

has been adapted to obtain a rectangular 5:1 cylinder. All details concerning the model and the experimental arrangement can be found in [Schewe \[1989\]](#). The model used in [Shirato et al. \[2010, 2011\]](#) is made of thin aluminum plate in its  $B/D=5$  main set-up, while an additional frame made of foamed styrol is inserted between one side surface aluminum plate and the remaining open box-shape model in order to investigate further  $B/D$  ratios.

As for measurement techniques and outputs, pressure taps placed on the model surface are used in [Bartoli et al. \[2011\]](#), [Bronkhorst et al. \[2011\]](#), [Shirato et al. \[2011\]](#) to provide the statistics of the pressure distribution at the body surface while unsteady aerodynamic loads are measured in [Schewe \[2009\]](#) by means of a high-stiffness piezoelectric balance. Experiments are carried out in [Schewe \[2009\]](#) for different angles of attack and for a wide range of Reynolds numbers ( $2 \cdot 10^4 \leq \text{Re}_D \leq 2 \cdot 10^6$ ). In the following, only the case at  $\alpha = 0$  will be considered and the values of the forces measured for  $\text{Re}_D = 26400$  will be used for comparison, as also done in [Mannini et al. \[2011\]](#), [Mannini and Schewe \[2011\]](#), while the effects of Reynolds number will be briefly discussed (see Sec. 4.1).

As suggested in the guidelines of the BARC benchmark, the contributions by [Bronkhorst et al. \[2011\]](#) and [Bartoli et al. \[2011\]](#) report calibration studies of the model and of the wind tunnel set-up, and both point out the difficulties associated with obtaining a perfectly symmetric configuration. In particular

Table 3

Wind tunnel tests: set-up characteristics

source	$\text{Re}_D$	$L/B$	Blockage (%)	$I_x$ %	$L_x/D$
<a href="#">Schewe [2009]</a>	26400	10.8	1.83	0.4	–
<a href="#">Bronkhorst et al. [2011]</a>	50000 - 100000	4	3.5	0.32-0.56	–
<a href="#">Bartoli et al. [2011]</a>	20000 -80000	7.93	3.75	1.6-2.1	1.27-1.87
<a href="#">Shirato et al. [2011]</a>	–	3,6	–	10.5,11.5,14	0.92-3.16

several causes of asymmetry in the experiment conditions are identified and investigated, the main ones being the disturbances in the incoming flow, misalignment of the model with the incoming flow and inaccuracies in the model geometry. These three causes are separately analysed in [Bronkhorst et al. \[2011\]](#). The two research teams use different approaches when aligning the model in the wind tunnel. [Bartoli et al. \[2011\]](#) use a trial and error approach, by rotating the model, horizontally placed in the wind tunnel, around its axis, and checking the value of the stagnation pressure coefficient. The tests are then carried out for the angle giving the largest value of the stagnation pressure coefficient, which turns out to be equal to 1. This approach allows compensating for possible flow non symmetries. [Bronkhorst et al. \[2011\]](#), on the other hand, align the model, vertically placed in the wind tunnel, such to have its faces perpendicular to the tunnel walls (a turntable permits an accuracy of 0.05). This configuration brings a stagnation pressure coefficient of 1. In spite of the attention paid to the alignment of the model, both studies show a clear non

symmetry in the mean and RMS pressure coefficients between the upper and lower faces. Conversely, [Schewe \[2009\]](#), for all the considered Reynolds numbers and  $\alpha = 0$ , obtains values of the mean lift coefficient practically equal to zero, which indicate a symmetry of the mean flow. Asymmetries in the mean flow have also been observed in some of the numerical contributions and this issue will be discussed in detail in Sec. 4.2.

### 3.2 Computational simulations

The various numerical contributions differ for physical modeling, numerical methods and simulation set-up. We refer to the original papers for a complete description, while an overview is herein given of those aspects which have been the object of extensive sensitivity studies by the contributors.

A first issue is clearly turbulence modeling. The different numerical studies cover a wide range of approaches to turbulence (see Tab. 4), even if the studies based on LES and DES prevail over the ones using Unsteady Reynolds Average (URANS) models, the latter being restricted to the works of [Mannini et al. \[2010, 2011\]](#) and [Ribeiro \[2011\]](#). LES simulations represent 51% of the numerical contributions, the DES ones 30% and, finally, URANS computations 29%. Nevertheless, a significant number of URANS models have

Table 4

Computational studies: turbulence modeling

source	$Re_D$	turbulence model	
		approach	closures
<a href="#">Arslan et al. [2011]</a>	$2.64 \times 10^4$	LES	SM, DSM, KET
<a href="#">Bruno et al. [2010, 2011]</a>	$4. \times 10^4$	LES	KET
<a href="#">Grozescu et al. [2011a,b]</a>	$2. \times 10^4, 4. \times 10^4$	VMS-LES	SM, WALE
<a href="#">Mannini et al. [2010, 2011];</a>	$2.64 \times 10^4, 10^5$	URANS	SA, LEA $k - \omega$
<a href="#">Mannini and Schewe [2011]</a>		DES	SA
<a href="#">Ribeiro [2011]</a>	$2.64 \times 10^4$	URANS	RSM, SST $k - \omega$ real. and RNG $k - \varepsilon$
<a href="#">Wei and Kareem [2011]</a>	$10^5$	LES	DSM
		IDDES	SA

been applied to the test case, thanks to the affordable computational cost of each simulation: 1-equation SpalartAllmaras model (SA), Linearized Explicit Algebraic Wilcox  $k - \omega$  model (LEA  $k - \omega$ ), Menter  $k - \omega$  model (SST  $k - \omega$ ), realisable and RNG  $k - \varepsilon$  models, Reynolds Stress Model (RSM).

Testing the performance of the URANS approach is surely useful in an engineering perspective, as industrial applications often require simple and cheap 2D URANS simulations to investigate a large number of flow parameters and geometry configurations [e.g. in [Ribeiro, 2011](#)]. As for LES, both the classical formulation and the Variational Multi Scale one (VMS-LES) are tested in conjunction with a number of sub-grid models: Standard and Dynamic Smagorinsky Model (SM and DSM, respectively), Kinetic Energy one-equation model (KET), Wall-Adapting Local Eddy-viscosity (WALE) model. Finally, as for hybrid methods, both classical DES and Improved Delayed Detached Eddy Simulation (IDDES) are employed, where the SA model is adopted in the URANS part of the model. A more detailed description and the precise references for the adopted turbulence approaches and models can be found in the papers cited in Tab. 4.

As for numerical discretization, commercial codes [Fluent in [Arslan et al., 2011](#), [Ribeiro, 2011](#)], opensource codes [Openfoam in [Bruno et al., 2011](#), [Wei and Kareem, 2011](#)] and proprietary codes [[Grozescu et al., 2011a,b](#), [Mannini et al., 2010, 2011](#), [Mannini and Schewe, 2011](#)] are used. Fluent and Openfoam are among the most widely used codes for CFD applications; the validation of the proprietary code used in [[Grozescu et al., 2011a,b](#)] for the simulation of bluff-body flows can be found e.g. in [Ouvrard et al. \[2010\]](#), [Wornom et al. \[2011\]](#), while the unstructured Tau code developed at the German Aersopace Center(DLR, Germany) and largely validated in the past (see e.g. [Gerhold et al. \[1997\]](#)) is adopted in [Mannini et al. \[2010, 2011\]](#), [Mannini and Schewe \[2011\]](#). All codes are based on the finite-volume method, except for the one used by [Grozescu et al. \[2011a\]](#), based on a mixed finite-element/finite-volume discretization. The adopted numerical methods are all second-order accurate in space and time. A crucial issue in LES and hybrid URANS/LES is the introduction of numerical dissipation, which is needed, in particular, to stabilize compressible flow solvers. The numerical viscosity in [Grozescu et al. \[2011a,b\]](#) is designed for LES and it has been shown in previous studies to have negligible effects on the quality of the results (see e.g. [Ouvrard et al. \[2010\]](#) and the references therein). The scalar Jameson-Schmidt-Turkel scalar artificial dissipation is used in [Mannini et al. \[2011\]](#), while a matrix artificial dissipation is employed in [Mannini and Schewe \[2011\]](#). [Mannini et al. \[2011\]](#) and [Mannini and Schewe \[2011\]](#) investigate the sensitivity of the results to the type and amount of artificial dissipation. No further sensitivity studies to numerics have been carried out.

As for the simulation set-up, the geometry of the spatial domain is characterized according to the BARC nomenclature given in [Bartoli et al. \[2008b\]](#) (Fig. 2). The values of the parameters adopted in each study are collected in Tab. 5. Two-dimensional domains are adopted in 30% of the simulations in conjunction with URANS models, except for a single LES simulation performed by [Arslan et al. \[2011\]](#) and aimed at comparing its accuracy with the 3D LES simulations in the same work. The shape of the domain is mostly prismatic (P) but cylindrical domains (O) are also generated by [Mannini and Schewe](#)

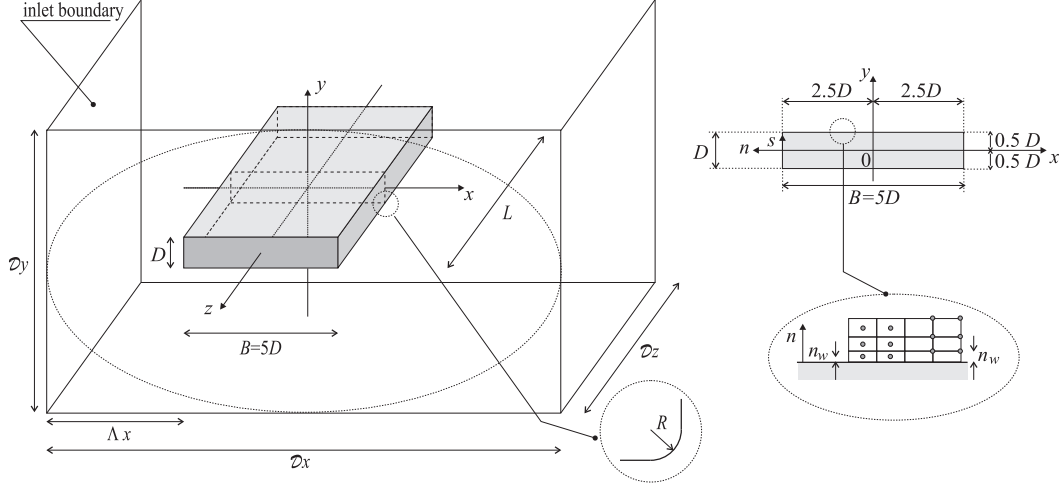


Fig. 2. Computational studies: model and domain geometry

Table 5

Computational studies: parameters of the computational domain

source	dim.	shape	$\mathcal{D}_x/B$	$\mathcal{D}_y/B$	$\mathcal{D}_z/B$	$\Lambda_x/B$	$R/D$
Arslan et al. [2011]	3D	P	23	20.2	1	10	0
	2D	P	23	20.2	—	10	0
Bruno et al. [2010, 2011]	3D	P	41	30.2	1,2,4	15	0
Grozescu et al. [2011a,b]	3D	P	41	30.2	1	15	0
Mannini et al. [2010, 2011];	2D	O	200	200	—	100	0
Mannini and Schewe [2011]	3D	O	200	200	1,2	100	0
Ribeiro [2011]	2D	O	51	50.2	—	25	0 — -0.1
Wei and Kareem [2011]	3D	P	8	3	0.2,0.4,1	1.5	0

[2011] and Ribeiro [2011], in conjunction with an O-grid in the latter study. No specific sensitivity studies are available in literature about the domain size for computational simulations of rectangular cylinders, nor such a study has been systematically accomplished during the BARC activity up to now, except for the spanwise dimension (Bruno et al. [2011, 2012]). Nevertheless, one may refer to the extensive parametrical studies by Posdziech and Grundmann [2007] and Qu et al. [2013] aimed at assessing the influence of the computational domain size on the simulated flow around an unconfined circular cylinder at low Reynolds numbers. The authors conclude that, in any case, the distance of the cylinder from the inlet boundary  $\Lambda_x$  and from each lateral boundaries  $\mathcal{D}_y/2$  should be larger than  $20D$ . In spite of the different geometry and Re number, these may be considered as first indications also for the BARC test case. In most cases, the domain dimensions adopted by the BARC participants in the  $x$  and  $y$  directions are about 20 to 50 times the breadth  $B$  of the cylinder section, with remarkable exceptions in Mannini and Schewe [2011] ( $\mathcal{D}_x/B = \mathcal{D}_y/B = 200$ ) and in Wei and Kareem [2011] ( $\mathcal{D}_x/B = 8$ ,  $\mathcal{D}_y/B = 3$ ): the former is expected to increase the computational cost but, of course, it avoids spurious effects of the boundary conditions on the simulated flowfield, while the domain size adopted in the latter is suspected to strongly affect the

flow. In particular, the blockage in the simulations in [Wei and Kareem \[2011\]](#) is of 6.7%, which is significantly larger than in the other simulations and in experiments and it does not fulfil the requirement of the benchmark of maximum 5% blockage (see Sec. 2). The very small distance of the cylinder leading edge from the inlet of the computational domain,  $\Lambda_x/B = 1.5$ , might also lead to undesired and uncontrolled perturbations of the incoming flow. The ensemble-average value of  $\mathcal{D}_z/B$  of the different contributions is close to one, but higher values are adopted in studies addressed to the evaluation of spanwise correlation [Bruno et al. \[2011, 2012\]](#), [Mannini et al. \[2011\]](#), while shorter lengths are adopted by [Wei and Kareem \[2011\]](#). The issue of the choice of the domain spanwise length will be analyzed in more detail in Sec. 4.

As for the cylinder geometry, perfectly sharp edges are adopted in all cases, while [Ribeiro \[2011\]](#) also varies the radius of curvature of the edges of the cylinder in a broad range.

Tables 6 and 7 summarize some features of the generated spatial grids. Most

Table 6

Computational studies: grid type

source	dim.	grid type	
		$x$ - $y$ plan	$z$ -wise
<a href="#">Arslan et al. [2011]</a>	3D	hybrid	struct.
	2D	hybrid	–
<a href="#">Bruno et al. [2010, 2011]</a>	3D	hybrid	struct.
<a href="#">Grozescu et al. [2011a,b]</a>	3D	unstr.	unstr.
<a href="#">Mannini et al. [2010, 2011];</a>	3D	hybrid	struct.
<a href="#">Mannini and Schewe [2011]</a>	2D	hybrid	–
<a href="#">Ribeiro [2011]</a>	2D	struct.	–
<a href="#">Wei and Kareem [2011]</a>	3D	struct.	struct.

of the grids are hybrid in the  $x - y$  plane (i.e. body-fitted, structured in the near wall region and unstructured triangular or quadrilateral elsewhere), and structured along the spanwise direction  $z$ . Remarkable exceptions are the fully unstructured grids adopted by [Grozescu et al. \[2011a\]](#) and the structured ones used by [Wei and Kareem \[2011\]](#) (orthogonal) and [Ribeiro \[2011\]](#) (non orthogonal). Table 7 compares the grid spacing normal to the wall,  $n_w$ , and in the  $x$ - and  $z$ - directions,  $\delta_x$  and  $\delta_z$ . Most of the grids adopts  $2. \times 10^{-5} \leq n_w/B \leq 5. \times 10^{-4}$  in order to obtain a grid resolution in wall units  $\overline{n}^+ \approx 1$ , and to fully resolve the boundary layer without introducing wall-functions-like approximations. A large scatter is observed in the values of  $\overline{\delta_x}/B$ : it is worth pointing out that the large value of  $\overline{\delta_x}/B$  in [Mannini and Schewe \[2011\]](#) is justified by the DES model, while the one of the coarsest grid in [Grozescu et al. \[2011a\]](#) has been conceived to test the VMS-LES

Table 7

Computational studies: grid resolution.  $\delta_x$  and  $\delta_z$  are the grid spacing in the  $x$  and  $z$  directions respectively, while  $n_w$  is the wall normal grid spacing and  $n^+$  is  $n_w$  made nondimensional in wall units.

source	$n_w/B$	$\overline{n^+}$	$\overline{\delta_x}/B$	$\delta_z/B$
Arslan et al. [2011]	$2. \times 10^{-4}$ 0.5	$4. \times 10^{-3}$	0.04	
Bruno et al. [2010, 2011]	$5. \times 10^{-4}$	1.66	$2. \times 10^{-3}$	0.042—0.01
Grozescu et al. [2011a,b]	$5. \times 10^{-4}$ , $2.5 \times 10^{-4}$	$\approx 1$	1.e-2, 5.e-3	0.042, 0.01
Mannini and Schewe [2011]	$5. \times 10^{-5}$	0.25	$1.4 \times 10^{-2}$	0.0156
Ribeiro [2011]	4.e-5	$\approx 1$	2.e-3	—
Wei and Kareem [2011]	n.a.	0.9—7.44	$10^{-2} - 3.3 \times 10^{-3}$	0.04—0.0025

accuracy in conjunction with very coarse grids. The spanwise grid resolution varies from  $\delta_z/B \approx 1/24$  in coarse grids to  $\delta_z/B \approx 1/100$  in refined grids, with the remarkable exception of an even smaller grid step adopted by Wei and Kareem [2011].

The overall number of grid cells,  $n_c$ , varies over 4 orders of magnitude among the studies (Fig. 3a), mainly depending on the approach to turbulence, the domain dimension and size, the grid type. Nevertheless, most of the models adopt an overall number of cells around one million (Fig. 3b). Only Bruno

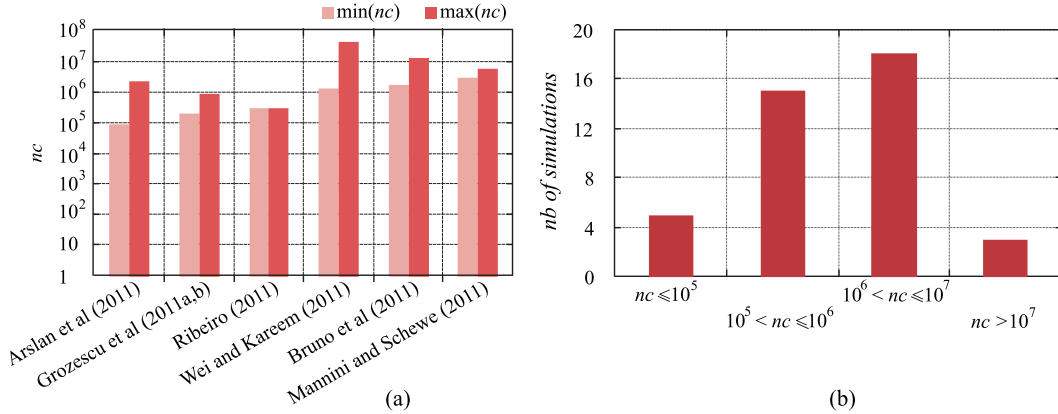


Fig. 3. Computational studies: number of cell for each study (a) and its overall distribution (b)

et al. [2010] provide some quantitative information about the cell skewness in the generated unstructured grid. Sensitivity studies to grid refinement are carried out in Bruno et al. [2011, 2012], Grozescu et al. [2011a,b], Mannini et al. [2010], Ribeiro [2011], Wei and Kareem [2011]. Grid independence has been shown only for the URANS simulations in Mannini et al. [2010].

As for the inflow conditions, the numerical contributions cover different values of freestream turbulence intensity and Reynolds number (see Tab. 4 and Fig. 1). Nonetheless, explicit sensitivity studies are carried out only for the Reynolds



number in Mannini et al. [2010], Grozescu et al. [2011a] and Grozescu et al. [2011b].

## 4 Comparison of the obtained results

For the sake of clarity, in the following the obtained results are schematically arranged in three subsections. The bulk parameters are compared in the first subsection, the main statistical quantities characterizing the flow and the aerodynamic load distribution over the cylinder surface are discussed in the second one, while the third one is devoted to pressure and force spanwise correlation. Because of the considered time-dependent flow, statistics in time of the above mentioned quantities are performed by all the authors. The convergence of the obtained statistical moments of each quantity versus the sampling time should be addressed in order to avoid systematic postprocessing errors. In particular, the extent of the sampling window is a key and critical element in the CFD approach, due to the computational costs involved in long physical time simulations. Most of the authors Arslan et al. [2011], Bronkhorst et al. [2011], Bruno et al. [2011], Grozescu et al. [2011a], Mannini et al. [2011], Mannini and Schewe [2011], Wei and Kareem [2011] check the convergence of the first and/or second order statistical moments of the bulk parameters. A possible lack of convergence is indicated only for the DES simulations in Mannini and Schewe [2011] and in some of the LES simulations of Wei and Kareem [2011]. The minimum extent of the sampling window which allows convergence residuals below a given threshold for a given statistical moment is quantitatively evaluated in some works Bronkhorst et al. [2011], Bruno et al. [2011], Grozescu et al. [2011a] by adopting the method proposed in Bruno et al. [2010]. The required sampling window depends on the simulated flow and hence on the experimental set-up and model parameters, on the quantity to be statistically studied (e.g., for the studied flow, statistics of the Strouhal number converge faster than the ones of the lift coefficient), and, finally, on the order of the statistical moment to be evaluated. Nevertheless, some rough and indicative values follows from studies: a sampling extent longer than 400 non-dimensional time units,  $tU/D$ , is recommended to evaluate first-order statistical moments (time-averaged values); at least 900 non dimensional time units are required to obtain reasonably converged values of the second-order statistical moments. However, as indicated in Mannini et al. [2011], URANS simulations, which resolve a smaller part of turbulent time fluctuations, may converge more rapidly.

#### 4.1 Bulk parameters

The main flow bulk parameters obtained in the different wind tunnel and numerical studies are reported in Tables 8 and 9:  $t - avg(C_D)$  and  $t - avg(C_L)$  are the time- and spanwise-averaged drag and lift coefficients per unit length, respectively;  $t - std(C_L)$  is the standard deviation of the time variation of the lift coefficient;  $St_D = f_s D/U$  is the Strouhal number, where the shedding frequency  $f_s$  is evaluated from the time fluctuations of the lift coefficient or from pressure or velocity time signals (we refer to the single cited articles for more details).

First of all, we remark that at present, among the different wind tunnel tests carried out in the framework of the BARC benchmark, bulk parameters are available only from Schewe [2006, 2009] and from Bartoli et al. [2011] (only the Strouhal number). In general, several wind tunnel data are available in the literature for the flow around the same body geometry as far as the Strouhal number is concerned and only a few for the mean drag coefficient; these data are also reported in Table 8 for comparison. Conversely, bulk-parameter values computed in 25-36 simulations of the BARC configuration are available. The histograms of the bulk parameters obtained by computational simulations are plotted in Fig. 4. For the sake of brevity, detailed values are not given herein (we refer to the cited papers) and only the range of the results obtained in all the simulations carried out in each single contribution is reported in Table 9, together with the ensemble average over all the available data, the standard deviation and the minimum and maximum deviations, these two last quantities being given in percent of the ensemble average.

The values of  $t - avg(C_D)$  obtained in most of the simulations are very close

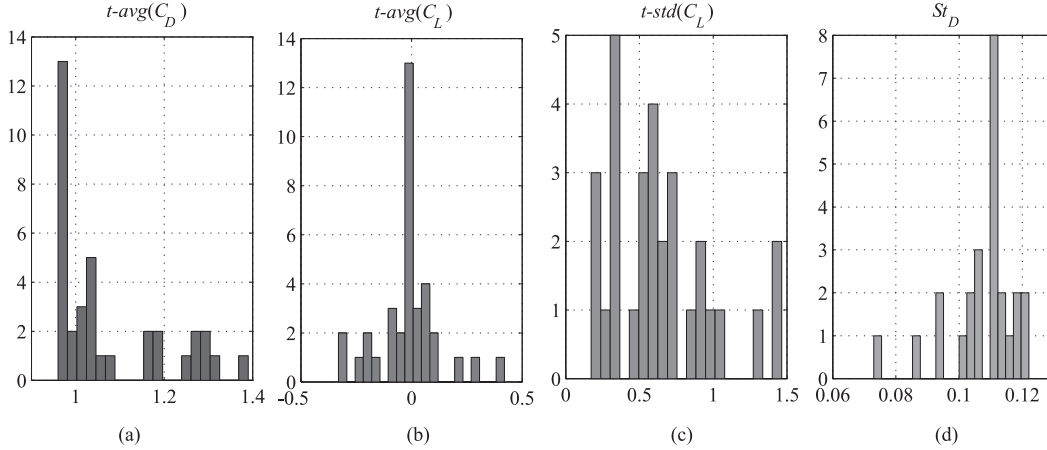


Fig. 4. Computational results: histograms of the bulk parameters; (a)  $t - avg(C_D)$  over 36 realizations, (b)  $t - avg(C_L)$  over 36 realizations, (c)  $t - std(C_L)$  over 30 realizations, (d)  $St$  over 25 realizations.

to 1 and this is in good agreement with the available wind tunnel data. In particular, Schewe [2009] obtains  $t - avg(C_D)$  equal to 1.029 at  $Re_D = 26400$ ,

while the measured values vary very little ( $0.94 \leq t - \text{avg}(C_D) \leq 1.05$ ) for  $2 \cdot 10^4 \leq \text{Re}_D \leq 2 \cdot 10^6$ . Moreover, an overall good agreement is observed between the predictions obtained in the various numerical studies, in spite of the previously outlined differences in numerics, modelling and simulation set-up. Indeed, the standard deviation of the data remains lower than 12% of the ensemble-averaged prediction. It will be shown in the following (Sec. 4.2) that, although the characteristics of flow on the cylinder sides significantly vary among the different simulations, the near-wake structure and, consequently, both the base mean pressure (see Fig. 5b) and the mean drag coefficient, show only small differences. Nonetheless, note that the maximum deviation of  $t - \text{avg}(C_D)$  from its ensemble average is rather large. The LES contributions to BARC give  $t - \text{avg}(C_D) \in [0.96, 1.39]$ , while the URANS and hybrid simulations give  $t - \text{avg}(C_D) \in [0.965, 1.295]$ . Therefore, rather surprisingly, the LES simulations are characterized by a larger dispersion than the URANS and hybrid ones, with a maximum discrepancy with the experimental value of [Schewe \[2009\]](#) of 35%. Nonetheless, it turns out that the largest values of  $t - \text{avg}(C_D)$  are obtained in the 2D simulation in [Arslan et al. \[2011\]](#) and in the LES carried out in [Wei and Kareem \[2011\]](#) on a small computational domain. If these simulations are discarded, the range of the LES predictions is reduced to  $t - \text{avg}(C_D) \in [0.96, 1.04]$ , with an ensemble average of 0.99 and the largest discrepancy from the experimental value of -10%. Therefore, it appears that the choice of the computational domain may significantly affect also the prediction of quantities which are rather unsensitive to modeling and to the other simulation parameters.

The Strouhal number is another quantity for which the predictions given in the different simulations are rather close to each other and in good agreement with the available wind tunnel data. Since  $St_D$  gives the dimensionless frequency of the vortex shedding behind the cylinder, this is a further confirmation that the dynamics of the near wake is satisfactorily captured in all the simulations in spite of the differences in the flow features on the cylinder lateral sides.

Conversely, the oscillations in time of the lift coefficient are very sensitive to the complex dynamics of the flow on the lateral cylinder sides. Indeed, a large spread of the numerical predictions is observed for the standard deviation of the lift coefficient. Note how the ensemble average of  $t - \text{std}(C_L)$  obtained in the different simulations is significantly larger than the only available wind tunnel value (Tab. 8). In this case, [Arslan et al. \[2011\]](#) does not observe a significant difference between 2D and 3D simulations. However, in general, this quantity seems to be sensitive to many numerical, modeling and simulation parameters. [Mannini and Schewe \[2011\]](#) show a significant impact of numerical dissipation, [Bruno et al. \[2011, 2012\]](#) and [Grozescu et al. \[2011a,b\]](#) point out a decrease of  $t - \text{std}(C_L)$  with increasing grid resolution, [Grozescu et al. \[2011a,b\]](#) a decrease with increasing Reynolds number and [Bruno et al. \[2011, 2012\]](#), [Mannini et al. \[2011\]](#) a decrease with increasing the spanwise extent of the computational domain. Finally, turbulence modeling has also a significant impact on the predictions of  $t - \text{std}(C_L)$ : lower values are obtained in [Mannini](#)

et al. [2011] in DES simulations than in URANS, while Wei and Kareem [2011] generally find lower values in LES than in IDDES. Note, however, that the IDDES predictions of  $t - std(C_L)$  of Wei and Kareem [2011] are significantly larger than those of the DES in Mannini et al. [2011]. Finally, Grozescu et al. [2011a,b] and Arslan et al. [2011] also observe remarkable effects of the SGS model in the predictions of this quantity obtained in LES simulations.

Finally, the mean lift coefficient is a-priori expected to be zero. Although values of  $t - avg(C_L)$  close to zero are obtained in most of the simulations, there are a few cases in which its absolute value is significant (Bruno et al. [2011, 2012] and Wei and Kareem [2011]). This might be due to the fact that the time interval used to compute the averaged quantities is not large enough to obtain statistical convergence. Nonetheless, in Bruno et al. [2011], a careful check of the convergence of the averaged quantities is made, and, hence, at least in that case, the statistical sample can be considered adequate. Therefore, it may be argued that a  $t - avg(C_L)$  value significantly different from zero is an indication of an asymmetry of the mean flow which may be triggered by very small perturbations of different nature. This point will be more deeply analysed in Sec. 4.2.

Table 8  
Bulk parameters: wind tunnel data

	$t - avg(C_D)$	$t - avg(C_L)$	$t - std(C_L)$	$St_D$
Bartoli et al. [2011]	–	–	–	0.12
Schewe [2006, 2009]	1.029	$\sim 0$	$\sim 0.4$	0.111
Nakamura and Mizota [1975]	$\sim 1$	–	–	–
Nakamura and Yoshimura [1982]	$\sim 1$	–	–	–
Nakamura and Nakshima [1986]	–	–	–	0.115
Nakamura et al. [1991]	–	–	–	0.118
Okajima [1982]	–	–	–	0.115
Okajima [1983] <sup>1</sup>	–	–	–	0.105
Parker and Welsh [1983]	–	–	–	0.105
Stokes and Welsh [1986]	–	–	–	0.105
Knisely [1990]	–	–	–	0.106
Matsumoto [2005] <sup>1</sup>	$\sim 1$	–	–	0.132
Ricciardelli and Marra [2008]	–	–	–	0.116

<sup>1</sup> Reported in Mannini et al. [2011]

Table 9

Bulk parameters: numerical results

	$t - avg(C_D)$	$t - avg(C_L)$	$t - std(C_L)$	$St_D$
Arslan et al. [2011]	0.984–1.39	–	0.59–0.84	0.07–0.16
Mannini et al. [2011]	0.968 – 1.071	0.0032–0.047	0.42–1.075	0.094–0.102
Mannini et al. [2010]	1.015–1.172	–	0.108–1.12	0.087–0.105
Mannini and Schewe [2011]	0.965–1.016	-0.087–0.085	0.173–0.553	0.087–0.119
Ribeiro [2011]	1.17	–	0.9	0.073
Grozescu et al. [2011b]	0.97–0.98	-0.097–0.0043	0.52–0.65	0.107–0.11
Grozescu et al. [2011a]	0.96	0.0022	0.35	0.122
Bruno et al. [2011] <sup>1</sup>	0.96 – 1.03	-0.315– -0.0024	0.2–0.73	0.112–0.122
Bruno et al. [2010]	1.03	–	0.73	0.112
Wei and Kareem [2011]	1.165–1.305	-0.33–0.42	0.495–1.465	–
Ensemble average	1.074	-0.0141	0.65	0.109
Maximum deviation: positive <sup>2</sup>	+ 29.4%	+156.5 %	+125.4%	+11.6%
Maximum deviation: negative <sup>2</sup>	-10.6 %	-122.6 %	-73.4%	-33.3%
Standard deviation	0.129	0.142	0.374	0.015
Shimada and Ishihara [2002]	0.975	–	0.03 – 0.12	0.103–0.119

<sup>1</sup> Also in Grozescu et al. [2011a]<sup>2</sup> In percent of the ensemble-average value

#### 4.2 Main flow features and statistics

The distribution of the pressure coefficient  $C_p$ , averaged in time ( $t - avg$  in the following), in the spanwise direction ( $z - avg$  in the following) and between the upper and lower half perimeters ( $side - avg$  in the following), is plotted in Fig. 5. Figure 5(a) reports wind tunnel measurements, while Figure 5(b) the computational results. The abscissa  $s$  denotes the distance from the cylinder leading edge measured along the cylinder side (see also Fig. 2). For the sake of completeness Figure 5(a) (and Fig. 8(a) in the following) also includes the data obtained in high turbulent incoming flows by Le et al. [2009], even if the experimental set-up significantly differs from the BARC main one. The data of Galli(2005) and Matsumoto(2005) are taken from Mannini et al. [2011].

As a first remark, the mean pressure values given by the different wind tunnel and computational contributions to BARC on the rear side of the cylinder,  $5.5 \leq s/D \leq 6$ , are very close to each other and in good agreement with the experimental data available in the literature (also reported in Fig. 5(a)), with the only exception of the RANS computation with the RSM model in Ribeiro [2011]. As already mentioned in Sec. 4.1, this leads to similar predictions of the time averaged drag.

Conversely, a significant spread of both wind tunnel and computational pressure values is observed in Fig. 5 on the lateral side of the cylinder. Figure 6 summarizes the ensemble statistics of the  $C_p$  distributions on the lateral cylinder side; in particular, the range of the wind tunnel and numerical  $C_p$  values is reported for 24 locations over the cylinder lateral side, together with

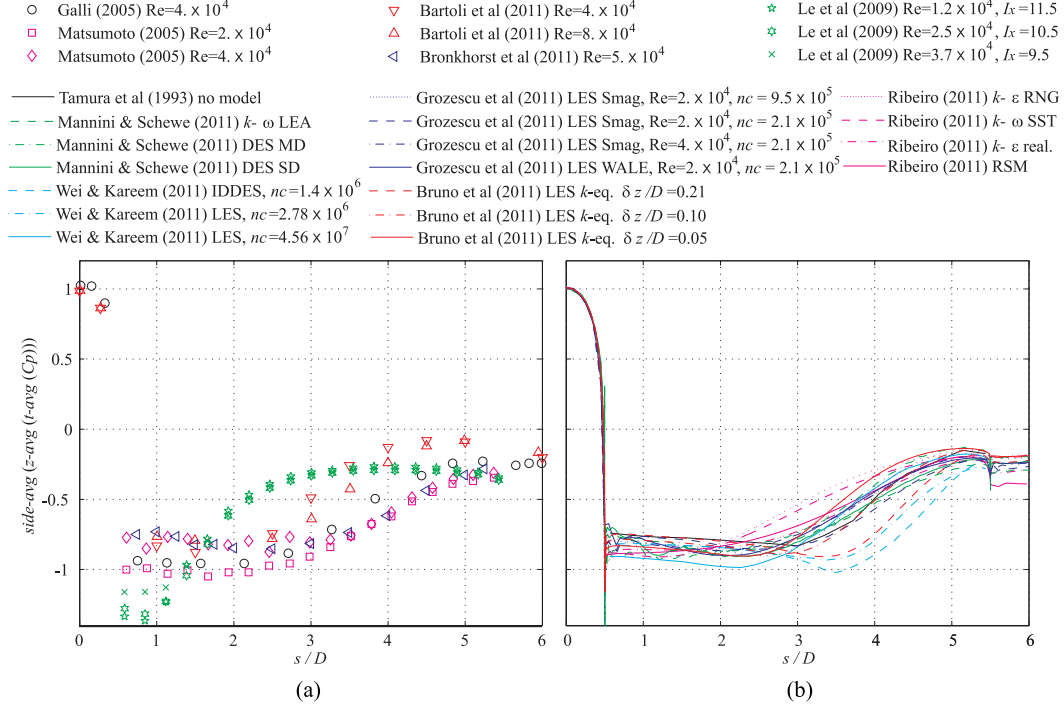


Fig. 5. Side-averaged, spanwise-averaged and time-averaged  $C_p$  distributions: wind tunnel (a) and computational (b) results

the median, the 25-th and the 75-th percentile values  $p_{25}$  and  $p_{75}$ , respectively, whiskers and outliers, if any. Points are drawn as outliers if they are larger than  $p_{75} + w(p_{75} - p_{25})$  or smaller than  $p_{25} - w(p_{75} - p_{25})$ ,  $w = 1.5$  being the maximum whisker length.

The subset of wind tunnel pressure data over which statistics are evaluated does not include the measurements obtained in high turbulent incoming flows Le et al. [2009], because such measurements do not belong to the same ensemble of smooth conditions. This holds also for the statistics of the standard deviation in time of the  $C_p$  (Fig. 9).

The results of Bartoli et al. [2011] are rather away from the remaining ones, which conversely well agree with each other. As for the numerical results, although most of the distributions are contained in a narrower range than the wind tunnel one (Figure 6), the shape of the mean pressure distribution significantly varies among the different simulations. The spread in the predictions of the mean pressure distribution is strictly linked with the differences in the mean flow topology obtained in the various simulations (see Fig. 7). Therefore, before analysing in more detail the behaviour of the mean pressure coefficient on the cylinder lateral size, the main features of the mean flow topology emerging from Fig. 7 are illustrated.

The mean flow on the cylinder lateral side is characterized by a main recirculation zone (or main vortex as in Bruno et al. [2010]), whose size and shape

significantly vary in the different contributions. To give a more precise quantification of this variability, Table 10 shows the  $x$  coordinate at which the mean recirculation zone ends (mean reattachment location  $x_r$ ) and the coordinates of the centre of the main recirculation zone. Only a subset of the contributions to BARC, for which these data were made available, is considered; as previously for the bulk coefficients, detailed values are not given in Table 10 for the sake of brevity (we refer to the cited papers), and only the range of the results obtained in all the simulations carried out in each single contribution is reported, together with the ensemble average and the standard deviation over all the available data. The value of  $x_r$  deduced by [Matsumoto et al. \[2003\]](#) from wind tunnel pressure measurements is also shown for reference. The data in Tab. 10 confirm a significant variability of the length and of the  $x$  position of the centre of the mean vortex, while the normal distance from the cylinder of its centre remains almost constant. This leads to very different shapes and curvature of the mean streamlines at the edge of this main recirculation zone. It is not easy to identify a trend with the different simulation parameters. In most of the numerical contributions to BARC (Fig. 7), a smaller recirculation is also visible very close to the lateral wall and immediately downstream of the upstream corner, which was already detected in [Bruno et al. \[2010\]](#) (see in particular Fig. 11 of [Bruno et al. \[2010\]](#), in which the mean flow structures are sketched). Again, its dimensions and shape vary significantly among the different simulations; in fact, in some cases it is hardly visible.

Coming back to the analysis of the mean  $C_p$  behaviour (Fig. 5), for all experiments and simulations adopting smooth and low turbulence incoming flow a first zone of almost constant low pressure can be observed; the length of the pressure coefficient plateau roughly corresponds to the distance from the upstream corner to the centre of the main vortex. In this first zone, rather surprisingly, the spread of the numerical results is lower than that of the wind tunnel ones. This may be due to the fact that in this upstream part of the lateral side, in spite of the previously analysed significant differences in the mean flow topology, the curvature of the mean streamlines is similar for all the simulations (Fig. 7). More downstream, in all cases the pressure increases again because of the change in the curvature of the mean streamlines as the mean flow tends to reattach (see Fig. 5). This is the region in which the largest spread among the different datasets is observed, in particular in the position at which the pressure starts to increase and in the steepness of the positive gradient (Fig. 5). Considering that the normal distance from the side surface to the centre of the vortex does not vary significantly (see Tab. 10 and the related discussion), in general a shorter recirculation zone leads to a larger curvature of the mean streamlines, which in turn implies a more rapid increase of the mean pressure coefficient.

The distribution of the standard deviation of the pressure coefficient, averaged also spanwise and between the upper and lower half perimeters, is plotted in Fig. 8. As an overall remark, a large scatter among the different experimental and numerical predictions of this quantity is evident. In this case the variabil-



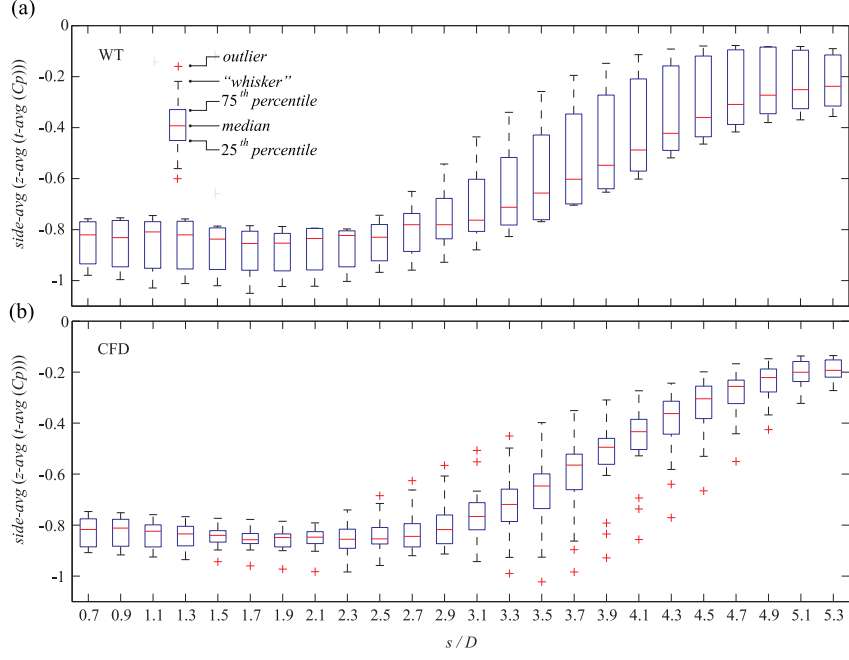


Fig. 6. Statistics of the side-averaged, spanwise-averaged and time-averaged  $C_p$  distributions: wind tunnel (a, 6 realisations) and computational (b, 18 realisations) results.

ity is larger for the numerical results than for the wind tunnel measurements, as also visualized in Fig. 9, in which the same statistics as in Fig. 6 are reported now for the  $C_p$  standard deviation. More particularly, on the upstream side of the cylinder the  $C_p$  standard deviation is very low in all the numerical simulations, while in most of the experiments is roughly around 0.05. This may be explained by the difference in the free-stream conditions, which are smooth in most of the numerical simulations, while turbulent fluctuations are present in the experiments, although the turbulence intensity is kept low. This is confirmed also by the fact that the data of [Le et al. \[2009\]](#), obtained for high levels of oncoming flow turbulence and reported in Fig. 8, show very large values of the  $C_p$  standard deviation in the upstream part of the cylinder side. As for its mean value, the largest variability of the  $C_p$  standard deviation is observed on the cylinder lateral side. In all cases there is a peak located slightly upstream the reattachment of the main mean recirculation vortex, in the zone where the mean  $C_p$  increases. In average, the peak is located more downstream and is more intense in numerical simulations than in experiments (compare Figs. 9a and 9b). Since the approach to turbulence is expected to have a significant impact on the numerical predictions of this quantity, Figs. 9c and 9d compare the distributions of the standard deviation of  $C_p$  obtained on the cylinder lateral surface in DES and LES simulations to those given by URANS models. Quite surprisingly the differences in the intensity and location of the main peak are rather small; therefore, it seems that turbulence modeling has an effect on the dynamics of the flow over the lateral cylinder sides which is comparable to that of other sources of uncertainties present in

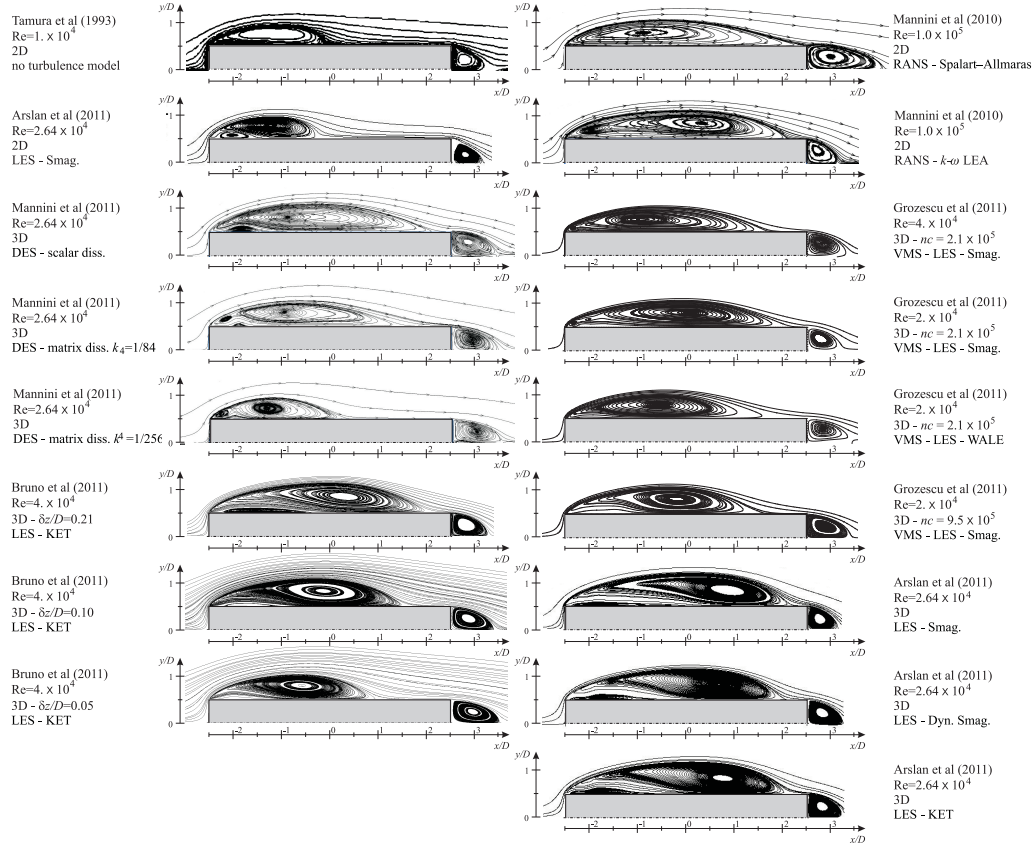


Fig. 7. Time and spanwise-averaged streamlines

the simulations and experiments. The main difference is that in DES and LES simulations, consistently with the wind tunnel measurements, the value of the standard deviation of  $C_p$  has an single peak along the cylinder side, while in the RANS ones a minimum is also found at a distance of approximately 2D from the upstream corner. The reasons for this behavior are not clear at this stage.

#### 4.2.1 Sensitivity to modelling and simulation parameters

Let us analyze herein in more detail the sensitivity of the numerical predictions of the pressure distribution and of the mean flow on the cylinder lateral surface to different modelling and simulation parameters, in order to possibly identify some trends or explanations of the result dispersion.

Let us consider first the mean pressure distribution, which has been previously shown to be linked to the mean flow topology. The approach to turbulence, viz. URANS, LES or hybrid RANS/LES, is expected to play a key role. However, none of these approaches seem to reduce the result dispersion, i.e. results significantly deviating from the ensemble average are found for LES, URANS and hybrid simulations.

Table 10

Mean reattachment location ( $x_r$ ) and coordinates of the centre of the mean recirculation zone ( $x_c, y_c$ )

	$x_r$	$x_c$	$y_c$
Arslan et al. [2011]	0.66 – –2.29	–1.24 – –0.75	0.78 – –0.82
Mannini et al. [2011]	2.25	–	–
Mannini et al. [2009] <sup>1</sup>	1.72 – –2.06	–1.44 – – – 0.05	0.77 – –0.88
Grozescu et al. [2011a]	1.65 – –2.1	–0.97 – –0.09	0.76 – –0.805
Grozescu et al. [2011b]	1.64	–0.17	0.35 0.82
Bruno et al. [2010]	2.18	0.04 <sup>1</sup>	0.8 <sup>1</sup>
Ensemble average	2.25	–0.23	0.804
Standard deviation	0.45	0.73	0.033
Matsumoto et al. [2003]	1.875	–	–

<sup>1</sup> Reported in Arslan et al. [2011]

<sup>2</sup> In percent of the ensemble-average value

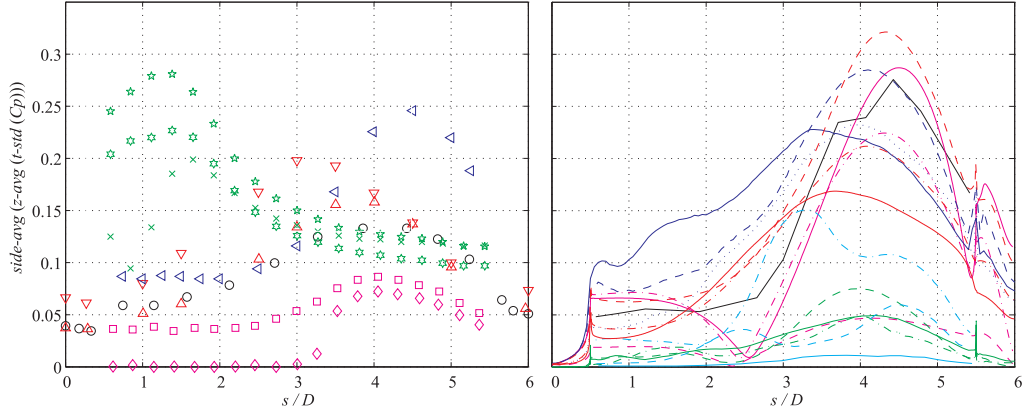


Fig. 8. Side-averaged, spanwise-averaged distributions of the standard deviation in time of the  $C_p$ : wind tunnel (a) and computational (b) results

For URANS, as expected, the largest sensitivity is to turbulence model; in particular, Ribeiro [2011] obtains a mean  $C_p$  distribution very close to the ensemble average for the realizable  $k-\varepsilon$  and the RSM models (the latter, however, giving a wrong prediction of the base pressure), while the distributions obtained with the RNG  $k-\varepsilon$  and the SST  $k-\omega$  models largely overestimate

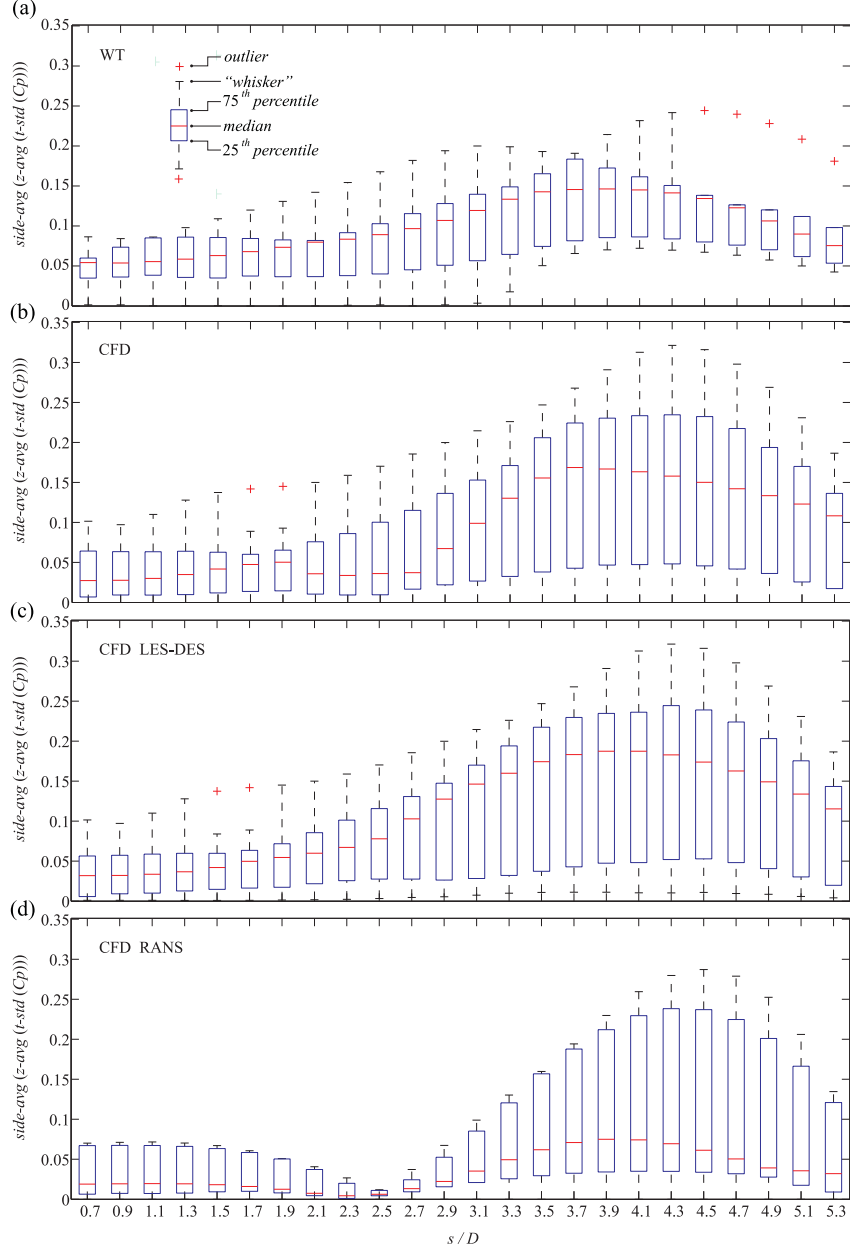


Fig. 9. Statistics of the side-averaged distributions of the standard deviation in time of the  $C_p$ : wind tunnel (a, 6 realisations) and computational (b, 18 realisations; c, 13 realisations; d, 5 realisations) results

the pressure increase in the downwind zone of the cylinder side. Finally, the URANS simulation of Mannini and Schewe [2011], which uses the LEA  $k - \omega$  model, provides a mean  $C_p$  distribution close to the ensemble average. For LES and hybrid URANS/LES approaches, the analysis is more complicated, since different parameters seem to have a comparable impact. As a first remark, it is pointed out once again in Arslan et al. [2011] that the 2D LES yields results significantly different from the 3D ones, and, in particular, a main recirculation zone much shorter than in 3D simulations. The effects

of SGS modeling in LES have been investigated in [Grozescu et al. \[2011a,b\]](#) and in [Arslan et al. \[2011\]](#). Both these studies show that the SGS model has a limited impact, in spite of the quite coarse grid resolution. As for the hybridization strategy in hybrid simulations, the mean  $C_p$  distributions obtained in the IDDES simulations of [Wei and Kareem \[2011\]](#) deviate much more from the ensemble average than those of the DES simulations in [Mannini and Schewe \[2011\]](#); this is probably due to the large difference in the adopted domain size adopted in the studies. The sensitivity to grid refinement has been investigated in LES simulations by [Grozescu et al. \[2011a,b\]](#), [Bruno et al. \[2011, 2012\]](#) and [Wei and Kareem \[2011\]](#); in [Grozescu et al. \[2011a,b\]](#) and [Wei and Kareem \[2011\]](#) the refinement is carried out in all directions, while [Bruno et al. \[2011, 2012\]](#) focus on refinement in the spanwise direction. All these studies agree on the qualitative behaviour, i.e. grid refinement leads to a decrease of the main vortex length and to an upstream displacement of its centre with the previously highlighted consequences on the pressure distribution. However, from a quantitative viewpoint, the effect of grid refinement is more limited in [Grozescu et al. \[2011a,b\]](#) than in [Bruno et al. \[2011, 2012\]](#) and [Wei and Kareem \[2011\]](#); this is rather surprising since the grid resolution in [Grozescu et al. \[2011a,b\]](#) is significantly coarser than in the other studies and it might be due to the use of unstructured grids, which allows more local refinements, and of the VMS-LES approach, which has been shown in previous studies to give more accurate results of classical LES on coarse grids (see e.g. [Ouvrard et al. \[2010\]](#), [Wornom et al. \[2011\]](#)). In particular, [Bruno et al. \[2011, 2012\]](#) show a striking impact of grid resolution in the spanwise direction; however, for the finest considered grid having  $\delta_z/D = 0.05$  a very short length of the main recirculation zone is obtained, close to that of the 2D LES in [Arslan et al. \[2011\]](#), and, consequently the mean pressure distribution becomes quite far from the ensemble average of the numerical predictions. The reasons of this behaviour are not clear at the moment and this point definitely deserves further investigation. As for the remaining simulation parameters, the impact of the spanwise length of the computational domain will be discussed in Sec. 4.3 dealing with pressure and force spanwise correlations. The sensitivity to the numerical method, and, in particular, to the numerical dissipation is investigated only in the DES simulations of [Mannini and Schewe \[2011\]](#), in which it is shown that the length of the main recirculation zone becomes smaller with decreasing numerical dissipation. Note, however, that the sensitivity is likely to depend on the specific used numerical scheme (see also the discussion in Sec. 3.2). The effect of the Reynolds number has finally been investigated in [Grozescu et al. \[2011a,b\]](#), in which it is shown that the main recirculation region becomes shorter as the Reynolds number is increased; however, the impact is definitely limited compared to the overall dispersion of the results.

Finally, the analysis of the distribution of pressure standard deviation is even more complex. In general, since the peak location is related to the length of the mean recirculation region, reference can be made to the discussion above.

As for the intensity of the peak and of the pressure fluctuations in general, differences in the results obtained by different turbulence approaches, which capture different amounts of turbulence fluctuations, can be expected; for instance, larger fluctuations are expected in LES simulations, especially those associated with very fine grid resolutions, than in URANS. The impact of the approach to turbulence has been previously discussed, pointing out that the observed behaviours do not completely fulfill the expectations and are not fully understood at present.

#### 4.2.2 Symmetry of the mean flow

Pressure coefficient distributions commented in the previous section result from the averaging between the upper and lower half perimeters, so that possible asymmetries with respect to the  $x$  axis can be cancelled. In fact, the pressure measurements in wind tunnel tests [Bartoli et al. \[2011\]](#), [Bronkhorst et al. \[2011\]](#) and some computational simulations [Bruno et al. \[2012\]](#), [Grozescu et al. \[2011a\]](#) have systematically, independently and explicitly discussed unexpected emerging differences in the statistics of pressure fields over the upper and lower lateral surfaces. The difference and the absolute difference of the  $t - avg(Cp)$  and  $t - std(Cp)$  distributions between the upper and lower half perimeters are shown in Figure 10 and Figure 11, respectively, for the cited studies. For a generic variable  $\phi$ , the difference is defined as  $\Delta\phi = \phi(s/D) - \phi(12 - s/D)$ , being  $0 \leq s/D \leq 6$ . In the same figure, also the corresponding boxplot statistic description is given. Significant differences between the upper and lower surfaces (up to  $\Delta(t - avg(Cp)) \approx 0.1$  and  $\Delta t - std(Cp) \approx 0.05$ ) are obtained in wind tunnel measurements and in some computational simulations. The distributions of the differences along the side surface show some common qualitative trends, i.e. they are not monotonic and they change sign along the central part of the lateral surface; in some cases, different authors obtain specular qualitative trend, i.e. a local maximum value of a distribution roughly corresponds to the minimum one in an other distribution and vice versa: compare for instance  $\Delta(t - avg(Cp))$  in [Bronkhorst et al. \[2011\]](#) and [Bartoli et al. \[2011\]](#), or in [Bruno et al. \[2012\]](#) and [Grozescu et al. \[2011a\]](#). As a consequence of these mirror-like, non monotonic distributions, the ensemble averaged value of the absolute differences has two maxima along the side surface at  $s/D \approx 2.5$  and  $s/D \approx 4.5$  (see fig. 11(d)), while the ensemble averaged value of the differences is close to zero at every location along the side surface, even if the number of realisations in the ensemble is quite low (see fig. 11(c)). The latter evidence recalls the theoretical result obtained by [Carassale \[2008\]](#), i.e. the symmetry of the ensemble-averaged flow around nominally symmetric set-up, also in case of non-ergodic phenomena. Nevertheless, a large number of realisations would be required to check the existence of a bistable flow and to confirm the convergence of the ensemble averaged value of the differences to zero. Nevertheless, these findings do not shed light about the cause(s) of the occur-

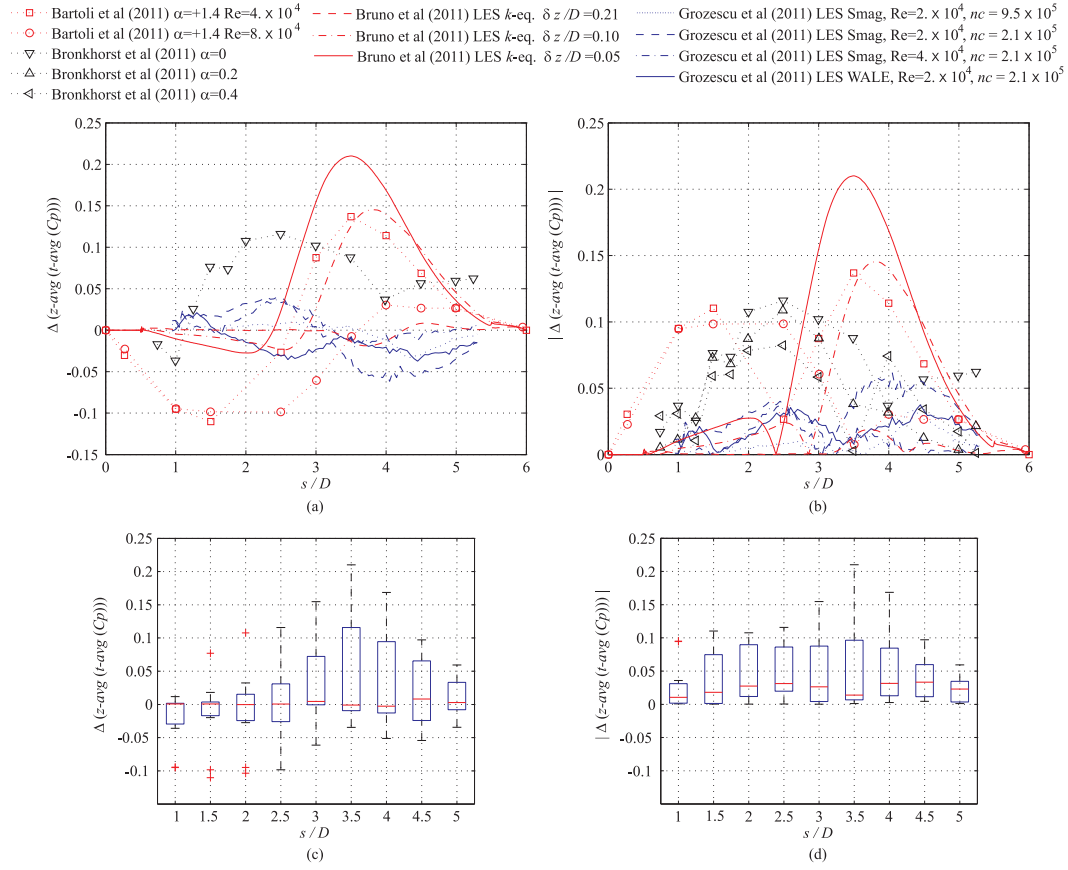


Fig. 10. Difference and absolute difference of the  $t - avg$  pressure coefficient distributions along at the upper and lower half perimeters: realisations (a, b) and corresponding statistics (c,d).

rence of single asymmetric flow realisations. Systematic errors due to insufficient sampling window in time and related lack of convergence in evaluating the pressure statistical moment are avoided by convergence assessment, as described at the beginning of sect. 4.

In the preliminary wind tunnel studies by [Bronkhorst et al. \[2011\]](#) and [Bartoli et al. \[2011\]](#) different possible causes of the non-symmetry were identified, the main being the disturbances in the incoming flow (e.g. by the upstream pitot-static tube), misalignment of the model with the incoming flow (e.g. misalignment of the model and/or asymmetry in the flow) and inaccuracies in the model geometry (e.g. degree of sharpness of the four edges, pressure taps disturbances and or quality of the rectangular shape of the model section). These three causes are separately and systematically addressed in [Bronkhorst et al. \[2011\]](#), while [Bartoli et al. \[2011\]](#) focus on last ones. Based on the performed investigations, [Bronkhorst et al. \[2011\]](#) conjecture that the asymmetry results from an inaccuracy in model shape, while misalignment of the relative flow is tentatively suggested by [Bartoli et al. \[2011\]](#). In spite of a number of trial re-alignments and model flippings, neither study yields a perfect symmetry of the time-averaged pressure field along the side surfaces.



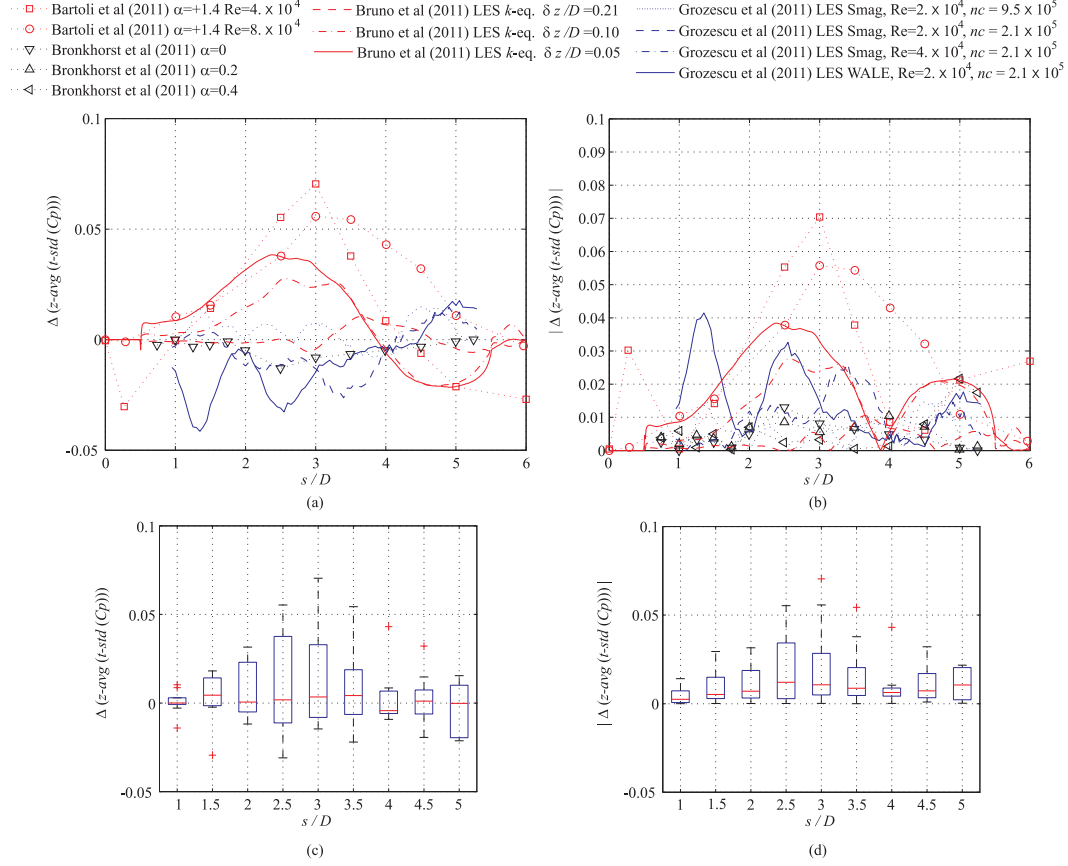


Fig. 11. Difference and absolute difference of the  $t - std$  pressure coefficient distributions along at the upper and lower half perimeters: realisations and corresponding statistics

The computational models are not affected by the wind tunnel inaccuracies cited above. In particular, the mathematical model accounts for perfectly rectangular cylinder section with perfectly sharp edges, while the laminar incoming flow and the other boundary conditions are fully symmetric. Hence, asymmetric flow can be triggered only by numerical issues. Local asymmetries exist in the spatial grid generated by Bruno et al. [2012] outside the grid boundary layer at the wall, while the mesh generated by Grozescu et al. [2011a] is fully symmetric with respect to the  $x - z$  plane. The procedures adopted for the numerical discretization and numerical solution of the governing equations (e.g. multigrid partition and parallelisation) can represent further potential causes of an asymmetric solution. The computational simulations performed by Grozescu et al. [2011a] provide smaller differences than the ones pointed out by Bruno et al. [2012]. Nevertheless, asymmetry in Grozescu et al. [2011a] is predicted by various subgrid models and holds at different Re numbers, even if a clear trend versus the latter can not be recognized. Conversely, the systematic grid refinement in the spanwise  $z$ -direction performed in Bruno et al. [2012] suggests that the asymmetric time and spanwise averaged pressure field can be simulated only by using a dense grid, while negligible differences take

place with the coarsest mesh. The authors conjecture that small coherent flow structures in the spanwise direction past the trailing edges are responsible of the whole asymmetric flowfield.

In general, the scrutinized studies show that the BARC flow has an high sensitivity to a number of factors which may trigger a significant asymmetry in the time-averaged flow field. It is worth recalling that some analogous well known examples of time-averaged asymmetric flows around nominally symmetric set-ups exist in bluff body aerodynamics, e.g. the circular cylinder in its critical regime [Bearman \[1969\]](#), [Schewe \[1983\]](#), or the side-by-side arrangement of two rectangular cylinders ( $B/D = 1.28$ ,  $S/D = 0.5$ , where  $S$  is the gap between cylinders) [Matsumoto et al. \[2008\]](#). Another example, which is more similar to the flow set-up discussed in this paper, is provided by [Ohya et al. \[1992\]](#), where the flow at  $Re_D = UD/\nu = 10^3$  around rectangular cylinders with different breadth to depth ratios ( $3 \leq B/D \leq 9$ ) is simulated by solving the 2D Navier-Stokes equations without turbulence models. The simulated time-averaged flow is symmetric for all the considered  $B/D$  ratios (included that one characterized by  $B/D = 5$ ), except for  $B/D = 4$ . It is worth noting that this asymmetry is not revealed in previous wind tunnel tests performed by the same authors [Nakamura et al. \[1991\]](#) under the same incoming flow nominal conditions.

Summarizing, some of the above mentioned causes of asymmetry are connected with the set-up parameters which are uncertain in both wind tunnel tests and computational simulations. Therefore, robust procedures to set, evaluate and control parametric uncertainties in wind tunnel test could be detailed in a Best Practice guideline for these type of studies, as proposed by [Bronkhorst et al. \[2011\]](#) and probabilistic approaches can be useful to account for the set-up uncertainties and to evaluate their propagation in the aerodynamic properties of bluff bodies, e.g. [Bruno and Fransos \[2011\]](#), [Han et al. \[2012\]](#). Further computational and wind tunnel investigations are also encouraged in the framework of the benchmark, e.g. addressed to evaluate the critical value of the set-up parameters driving the instability or to show the existence of two post critical asymmetric solutions of the bi-stable flow induced by disturbances of opposite sign.

#### *4.3 Pressure and force spanwise correlation*

Knowledge of the spanwise correlation is necessary to accurately evaluate the overall wind loads on real structures and of their response. Therefore, the comparative analysis of the spanwise correlation values obtained in the benchmark by means of different approaches and by different authors is a valuable complement to the results discussed in the previous sections. In particular, the accurate simulation of the flow spanwise features is a very demanding task and a challenging benchmark for the computational approach, mainly because of the requirements concerning the spanwise domain size and grid density. While

several studies have been devoted to the evaluation of the spanwise correlation in the case of the square cylinder [e.g. in McLean and Gartshore, 1992, Namiranian and Gartshore, 1988, Oka and Ishihara, 2009, Tamura et al., 1998, Vickery, 1966], the BARC benchmark gives a complement to a single study Ricciardelli [2010] already available in the literature about the spanwise correlation in the case of the high Re flow around the rectangular 5:1 cylinder. In what follows, the spanwise correlation of a generic aerodynamic variable  $C_i$  is described through its correlation coefficient

$$R_{C_i}(\Delta z/D) = \text{cov}[C_i(\Delta z/D)] / \text{var}[C_i] \quad (1)$$

where  $\Delta z/D$  is the spanwise separation between two points at which the variable is measured.

The pressure correlation coefficients available from both wind tunnel tests De Grenet and Ricciardelli [2005], Ricciardelli and Marra [2008] and computational simulations Bruno et al. [2012], Mannini et al. [2011] along three spanwise alignments  $s_i$ ,  $1 \leq i \leq 3$  are compared in Figure 12 and their statistics are given in terms of boxplots. The alignments are selected in order to

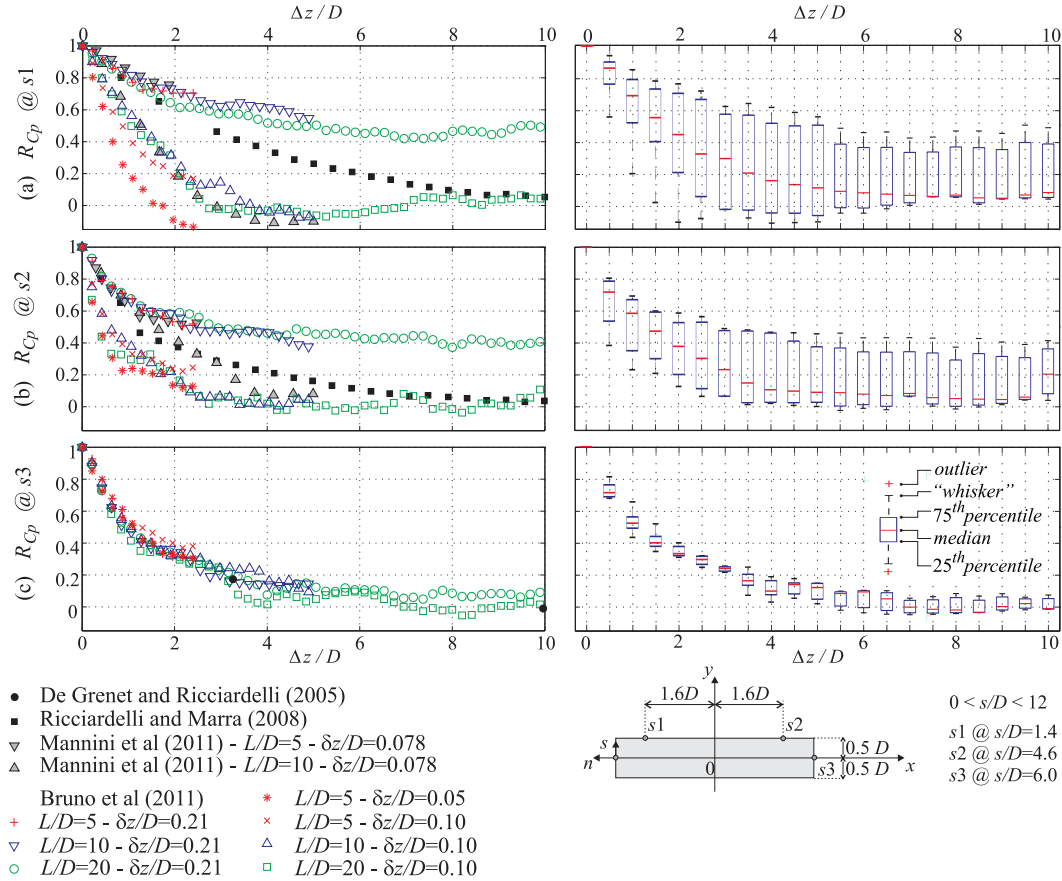


Fig. 12. Pressure correlation coefficients along three spanwise alignments: realisations and corresponding statistics

be representative of different flow regions, i.e. just downstream the separation

point in the recirculation bubble ( $s1$ ), close to the reattachment point ( $s2$ ), and on the rear side ( $s3$ ). The correlation coefficients of the lift and drag forces and their ensemble statistics are plotted in Figure 13. In particular, the lift correlation data include the wind tunnel measurements obtained by Shirato et al. [2010] under turbulent incoming flow characterized by a vertical turbulence intensity of about 11% and by different values of the vertical scale of turbulence  $L_y$  (for the other turbulence properties, see the cited paper). The pressure correlations at the rear surface (alignment  $s3$ ) and the

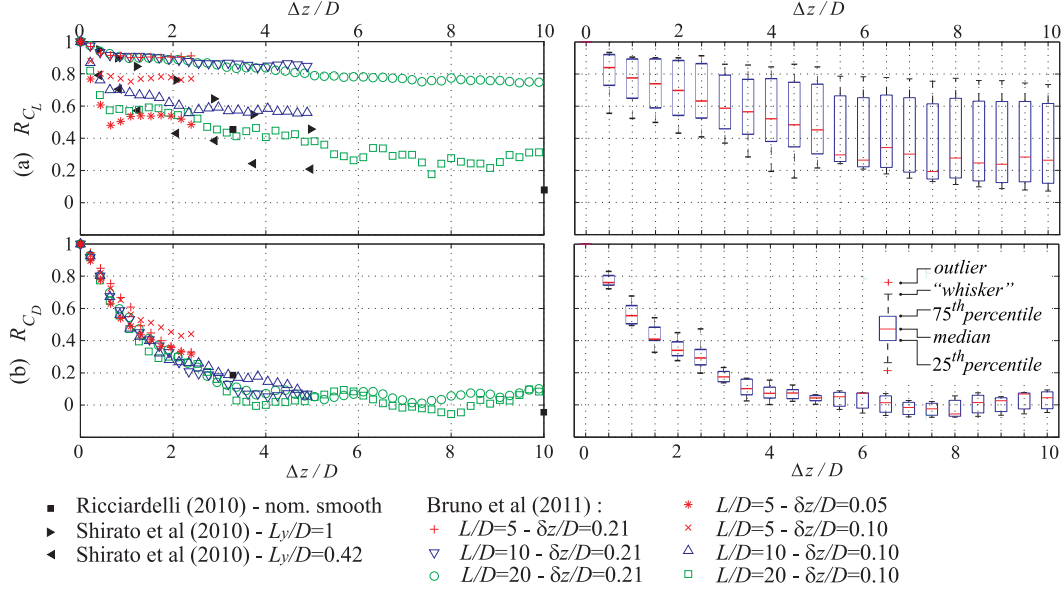


Fig. 13. Lift and drag correlation coefficients: realisations and corresponding statistics

drag correlations obtained by different approaches are in very good agreement among themselves, so that the ensemble of the results is characterized by a negligible deviation. Conversely, the collected data of the pressure correlation coefficient along the side surface (alignments  $s1, s2$ ) are significantly dispersed. The same statistical trend holds for the data of the lift correlation coefficient, which mainly results from the pressure distribution of the side surface. This dispersion confirms the sensitivity of the 3D flow along the side surface to a number of model parameters (e.g. the spanwise grid density  $\delta z$  and the domain length  $L$  in computational simulations) and incoming flow conditions (e.g. the turbulence length scale in wind tunnel tests).

Looking at the convergence study in Bruno et al. [2012], the spanwise grid spacing seems to play a key role in the simulation of the pressure correlation along the side surface, while no significant effects are shown by varying the cylinder length for a given grid step. Conversely, the results in Mannini et al. [2011] show a significant effect of the cylinder length for a given grid spacing ( $\delta z/D = 0.078$ ). However, in Mannini and Schewe [2011] it is stated that, when a less dissipative numerical scheme allowing a better resolution of span-

wise structures is used, the spanwise correlation of the pressure fluctuations significantly decreases. The reasons of these different sensitivities is not clear, even if can be ascribed to the different approaches to turbulence, numerical methods and to the grid spacing in the  $x - y$  plane. In spite of these different trends, some lessons can be learned: i. the coarsest grid step ( $\delta z/D = 0.21$ ) adopted in Bruno et al. [2012] is completely inadequate, regardless the cylinder length; ii. both studies seem to obtain the same pressure correlation along all the alignments for  $L/D \geq 10$  and  $0.078 \leq \delta z/D \leq 0.10$ . The latter correlation coefficient distribution does not seem to be the asymptotic one along  $s1$  looking at the results obtained by Bruno et al. [2012] with  $L/D = 5$  and  $\delta z/D = 0.05$ , but conclusive remarks cannot be drawn because: i. the lack of further simulations with longer cylinder lengths; ii. the overall flow simulated in Bruno et al. [2012] by adopting  $\delta z/D = 0.05$  is quite different than in the other simulations (see also Figure 7 and related comments). Finally, it is worth noting that the correlation coefficient predicted by LES and DES simulations along the alignment  $s1$  takes negative values for  $2 \leq \Delta z/D \leq 7$  before vanishing for larger values of the spanwise separation: a non monotonic trend of the correlation coefficient versus  $\Delta z$  follows. A sound phenomenological interpretation of this result has not been provided in Bruno et al. [2012], Mannini et al. [2011], and further investigations are welcome in the BARC future activities.

## 5 Conclusions

An overview has been presented aimed at summarising and disseminating the research activity performed during the first four year of activity of the Benchmark on the Aerodynamics of a Rectangular 5:1 Cylinder. In this overview, the BARC flow has been first put in context of the scientific literature available at the benchmark launching. Then, wind tunnel measurements and the computational simulations provided by 10 research teams have been compared. The results obtained from nominally common set-ups have been collected in a single ensemble of realisations in order to obtain statistics of several flow quantities, such as bulk parameters, chordwise pressure distributions, spanwise correlation. In spite of the common framework, the various contributions are characterized by remarkable differences, in set-up, experimental techniques, numerical methods and modeling parameters. Nonetheless, wind tunnel measurements and numerical predictions of base pressure and, hence, of the drag have been found to be in overall very good agreement. The same is for the Strouhal number. Conversely, standard deviation of the lift coefficient, the distribution of mean pressure and fluctuating pressure and the mean flow features on the cylinder side show a significant dispersion, both in experiments and simulations. Note that the uncertainty sources in numerical

simulations are different than in experiments, but all produce similar dispersion. This confirms the difficulty in identifying a reference experiment or simulation for the considered flow configuration, at least at present. Therefore, ensemble statistics over a sufficiently large number of realisations have been used to characterize at least some of the flow properties and of the aerodynamic loads. Nonetheless, the sensitivity to different parameters has been investigated for numerical contributions and some concluding remarks can be drawn. None of the approaches to turbulence seem to reduce the result dispersion, i.e. results significantly deviating from the ensemble average are found for LES, URANS and hybrid simulations. The results of URANS simulations are, as expected, mainly sensitive to the turbulence model, while in LES the impact of the SGS model appears to be lower than or similar to that of other simulation parameters. For example, the size of the computational domain seems to be a key issue; moreover, in a single contribution a striking sensitivity to the grid resolution in the spanwise direction has been pointed out, but the results obtained for the finest grid significantly deviate from the ensemble average of those of the experimental and numerical contributions. This point deserves therefore further investigation. The spanwise correlations of pressure and of aerodynamic forces were also investigated. The spanwise pressure correlation coefficients are more dispersed when evaluated on the lateral surface just downstream the separation point. It follows that the lift correlation coefficient is more dispersed than the drag one. Finally, an asymmetry of the time-averaged flow has been recognized in both preliminary wind tunnel tests and in computational simulations. This may be again explained by the extreme sensitivity of the flow to small uncontrolled uncertainties which can, in some cases, trigger the asymmetry of the mean flow.

The BARC activity is planned to continue up to 2015 under the umbrella of the International Association for Wind Engineering and in collaboration with the European Research Community On Flow, Turbulence And Combustion. In this framework, the authors hope to see the number of the flow realisations in the ensemble increased, in order to allow more precise statistics to be obtained including high-order moments and percentiles. In particular, collaborative studies combining wind tunnel tests and computational simulations are encouraged, as well as systematic sensitivity analyses to different parameters. The use of tools developed in the context of uncertainty quantification might also be useful to characterize the result variability with the different parameters. The complete database of the ensemble and its statistics will be made available online from the BARC website to the interested scientific and technical communities. Such a database may allow the performance of a measurement/simulation to be put and evaluated in a probabilistic context, or the results obtained from probabilistic models, such as e.g. those used for uncertainty quantification, to be directly compared in probabilistic terms. Furthermore, the BARC test case could be ported on the ERCOFTAC QNET-CFD Knowledge Base Wiki. From a fundamental viewpoint, a deeper analysis of the averaged flow asymmetry and of its causes is encouraged. Fi-

nally, a Best Practice Advice could be developed for wind tunnel tests and computational simulations of these kind of flow.

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